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*The Journal of Gear Manufacturing*

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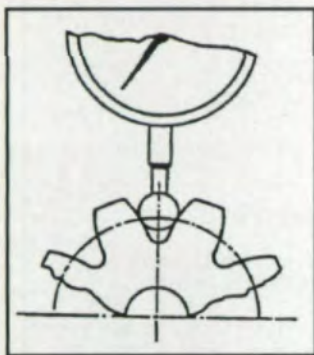
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# GEAR TECHNOLOGY

NOVEMBER/DECEMBER 1998

The Journal of Gear Manufacturing

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### RANDALL PUBLISHING STAFF

**President** Michael Goldstein

**Vice President** Richard Goldstein

**Controller** Patrick Nash

**Accounting** Laura Manion

**Art Consultant** Marsha Goldstein

**Phone:** 847-437-6604

**e-mail:** [people@geartechnology.com](mailto:people@geartechnology.com)

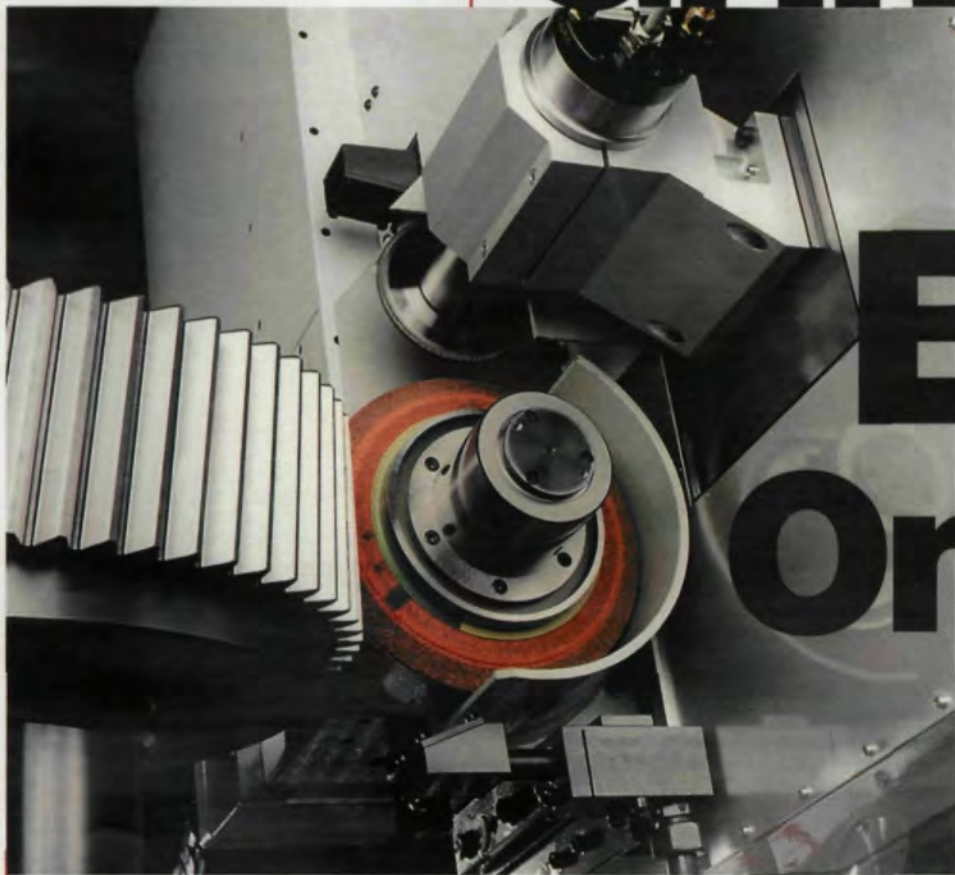


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CIRCLE 103

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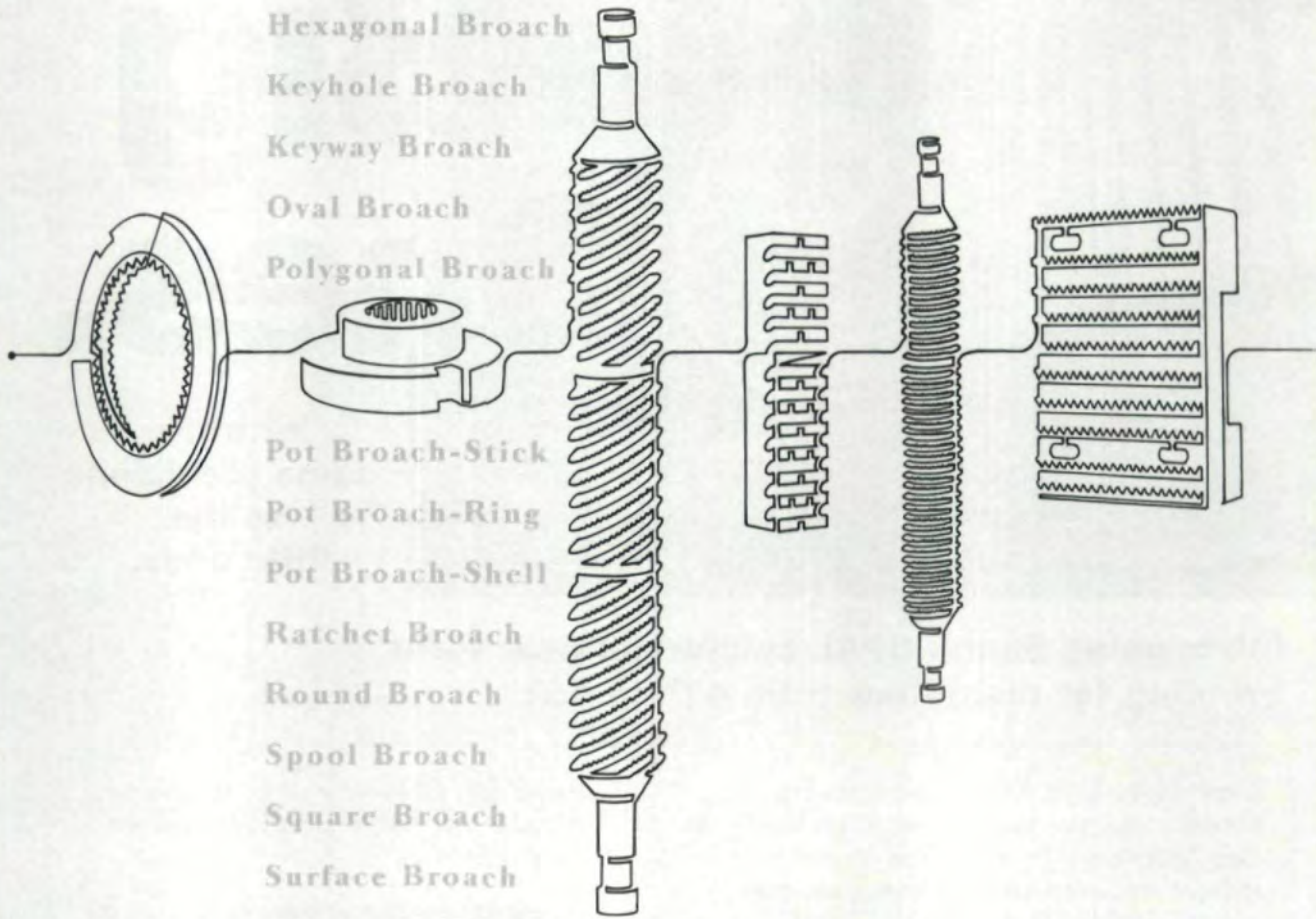
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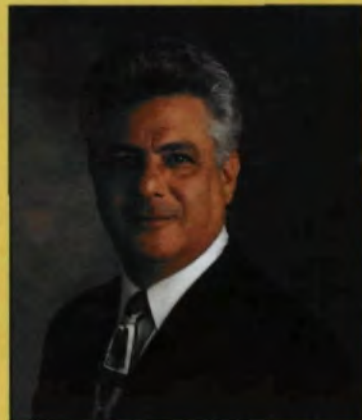
The carnival that is IMTS has come and gone. The aisles have been swept, and all the banners have been taken down. The fanfare of what some call the greatest machine tool show on earth has faded away.

We went to IMTS with the hopes of gaining insight into the health of both the economy and the gear manufacturing industry in America. Prior to the show, economy watchers around the world were getting skittish because of currency gyrations, political controversy and stock market fluctuations. Two months later, there is still much uncertainty. While America seems to be stable, we don't know how long it's going to last. We visited IMTS as much to have our fortune told as we did for the technological marvels of the show itself.

In many ways, IMTS lived up to its hype. It was the biggest IMTS ever. Attendance of 121,764 broke the previous high set in 1996. There were more exhibitors than ever before, filling the newly expanded McCormick Place's 1.4 million square feet to capacity. In addition, we saw the gear industry's latest technology, including 15 new gear machine models on display.

One of the most innovative machines was the new Liebherr-EMAG LCV 40 gear hobbing machine, which is built in a way that appears upside-down to someone who has spent his life around hobbing machines. The workpiece is driven from above, allowing for unobstructed chip removal below. The machine also uses a double work-spindle arrangement for automatic loading and unloading from a conveyor (one spindle is in cutting position while the other exchanges the cut gear for a new blank).

Gleason-Pfauter presented their first joint production line of machines. The GP series expands on the idea of modular machine building that has been used by other machinery manufacturers over the past several years and seems to be the direction of the future. The machines feature a common, modular base that can be equipped for hobbing, shaping or grinding, allowing the gear machine manufacturer a much faster delivery time, easier product planning and lessened inventory costs, which might translate to lower or fixed prices in the future.



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Below: The UG-coated cutting tools from National Broach were one of the many new technologies at IMTS.



We also were pleased to see several technological advancements in cutting tools. Both Pfauter-Maag Cutting Tools and Mitsubishi demonstrated dry cutting with high speed steel hobs rather than the brittle, expensive carbide hobs normally used. Both companies have worked to develop new steel alloys and proprietary coatings to enable dry cutting.

National Broach has come up with a way to save money on tool coating by developing a process that allows the tools to be resharpened without recoating. The new line of UG coated hobs and shaper cutters are coated with multiple layers of titanium nitride and titanium carbide as well as their new proprietary UG coating.

I'm sure you'll see and hear more about these products from our advertisers, for those of you who were unable to attend IMTS.

Despite all the positives that came out of the show, there was still a persistent feeling of uncertainty among the exhibitors. Many were less than enthusiastic about the quality and quantity of the traffic they saw at the show. In fact, the final attendance numbers surprised some who thought numbers might have been down by as much as 20-25%. All the exhibitors went home with a handful of sales leads, but nearly everyone's attitude was "let's wait and see if any of these pan out."

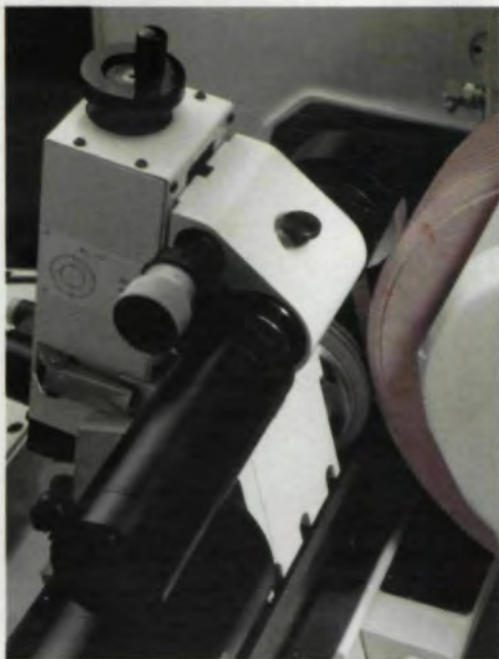
Immediate sales at a show like IMTS are hard to come by. But one success story we heard came from Star Cutter Co., who sold a CNC sharpening machine right off the show floor and who is currently negotiating the sale of a CNC automatic broach sharpener as a direct result of a contact made at the show.

Carnival fortune tellers earn their livings by making predictions that are bound to come true. The good ones make their predictions so general that any number of eventualities could prove them right. I regret to say that after visiting the show, I'm no more able to give you a clear vision of the future than a sideshow tarot reader.

We saw and heard much at IMTS to give us an optimistic outlook on our industry's chances for the next couple of years. But at the same time, the caution evident before the show doesn't seem to have gone away.

Caution isn't necessarily bad. When things are going well, we have more to look out for. The stakes are higher. In economic times like these, it's hard to ignore the headlines. But we know from our sources that sales of gear cutting tools are still very strong. We also know that at least one gear machine tool manufacturer has already sold nearly to capacity for 1999, and another company is raising its build forecast for '99 by 20%. This tells me that a lot of teeth are still being cut in America and that, for at least the near future, the American gear industry seems solid.

Michael Goldstein,  
Publisher and Editor-in-Chief



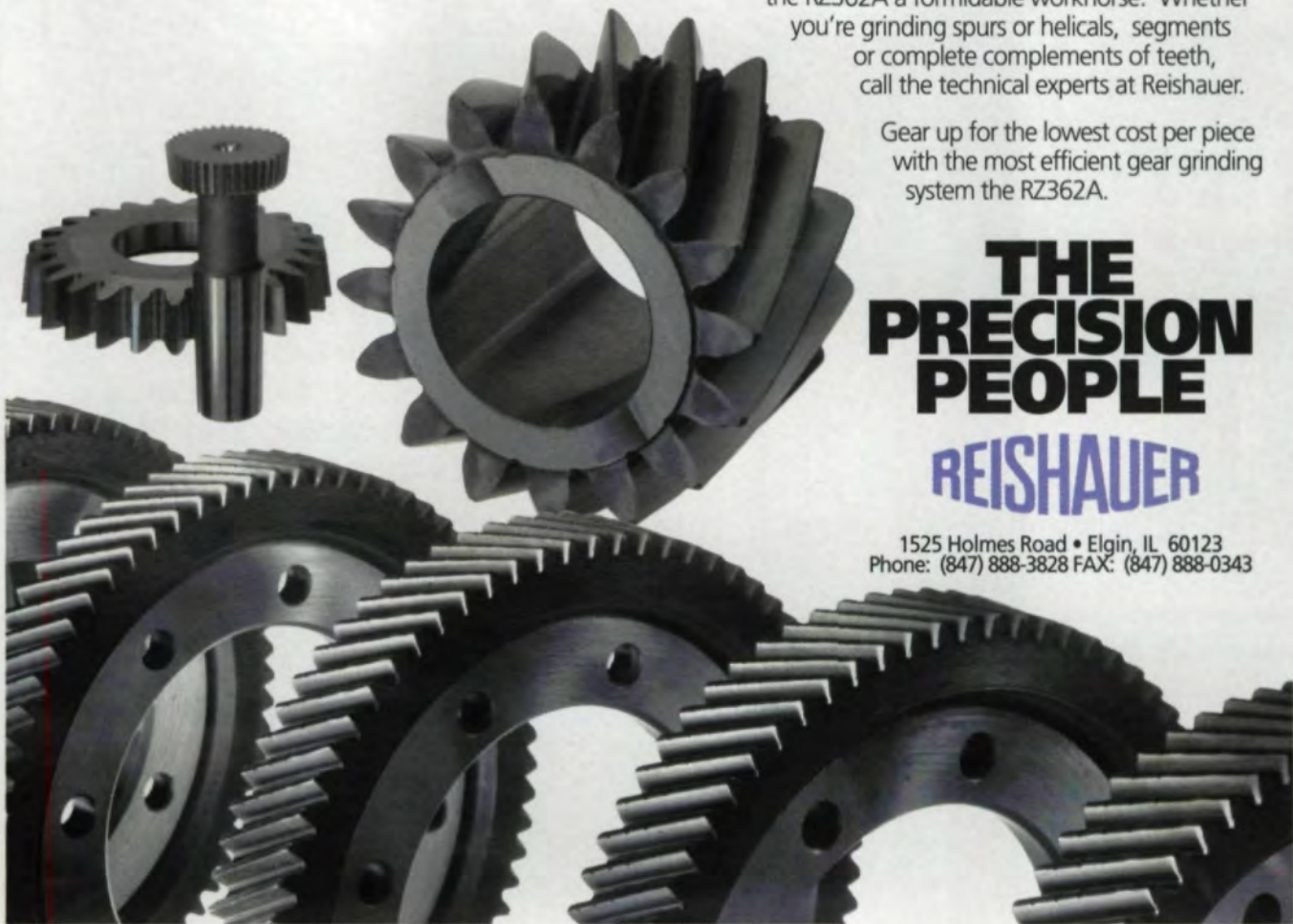
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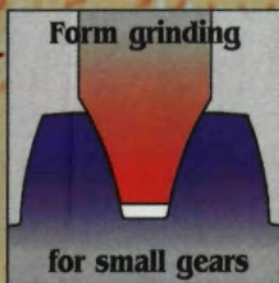


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CIRCLE 112

# Calculating Spur and Helical Gear Capacity with ISO 6336

Surface compressive (pitting) and tooth bending strength using ISO 6336-2;  
Calculation of surface durability (pitting) using ISO 6336-3; Calculation of tooth bending strength

Don McVittie

This is the third article in a series exploring the new ISO 6336 gear rating standard and its methods of calculation. The opinions expressed herein are those of the author as an individual. They do not represent the opinions of any organization of which he is a member.

## Pitting strength

ISO 6336-2 shows how to calculate the maximum contact stress and the permissible contact stress. It is based on the same Hertzian surface compressive stress theory as AGMA 2001, so we should expect similar equations. The fundamental ISO contact stress equations are:

$$\sigma_H = Z_B \sigma_{HO} \sqrt{K_A K_V K_{H\beta} K_{H\alpha}} \leq \sigma_{HP} \quad \text{ISO 6336-2. (1)}$$

$$\sigma_{HO} = Z_H Z_E Z_\epsilon Z_\beta \sqrt{\frac{F_t}{d_1 b} \frac{u+1}{u}} \quad \text{ISO 6336-2. (2)}$$

$$\sigma_{HP} = \frac{\sigma_{H \lim} Z_{NT}}{S_{H \min}} Z_L Z_V Z_R Z_W Z_X \quad \text{ISO 6336-2. (3)}$$

where:

$\sigma_H$  is the calculated contact stress, similar to  $s_c$  in AGMA 2001.

$\sigma_{HO}$  is the nominal contact stress.

$\sigma_{HP}$  is the permissible contact stress.

$\sigma_{H \lim}$  is the allowable contact stress number from ISO 6336-5, similar to  $s_{ac}$  in AGMA 2001.

The ISO 6336-2 equations look quite different at first, but they are similar to the AGMA equations. The terms  $Z_B$ ,  $Z_{HP}$ ,  $Z_E$ ,  $Z_\epsilon$ ,  $Z_\beta$  and  $(u+1)/u$  serve the same purpose as the AGMA  $I$  factor. When combined, they are nearly identical to  $\sqrt{I}$ . The differences are:

- $Z_B$ , which moves the calculated stress point from the pitch point to the lowest point of single tooth contact for spur gears, is not included in the nominal stress equa-

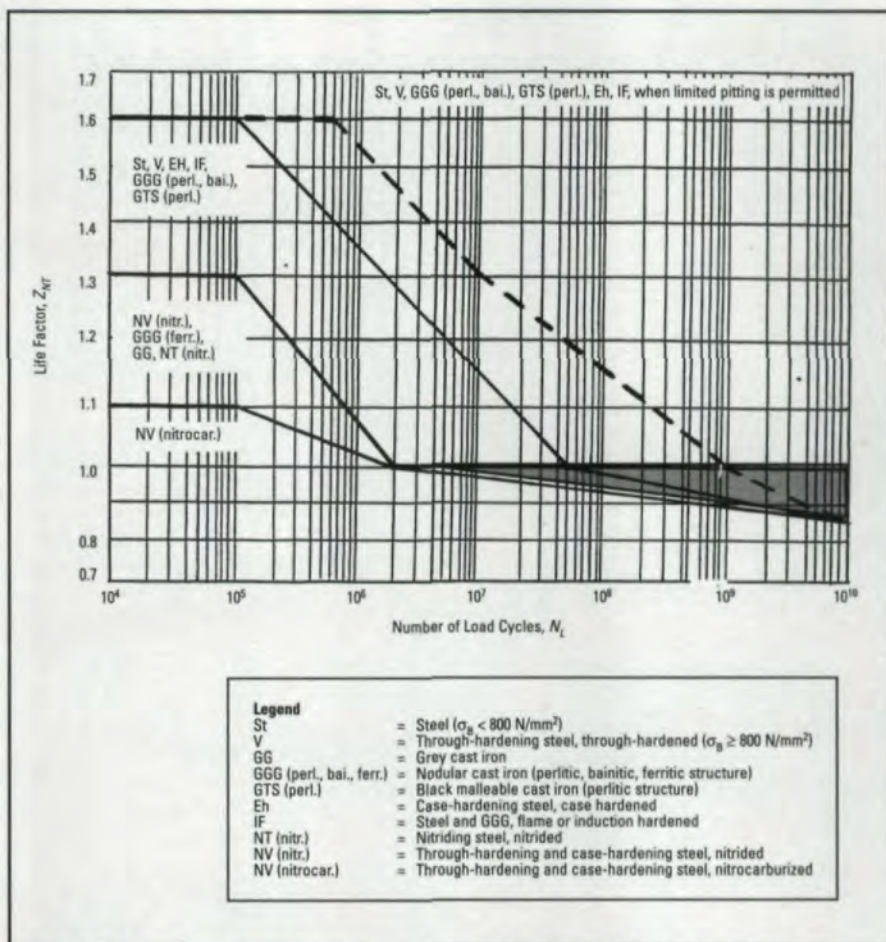


Fig. 1 — Life factor  $Z_{NT}$  for standard reference test gears, based on ISO 6336-2.

tion.  $Z_B$  is equal to 1.0 for helical gears.

- The value of  $Z_\epsilon$  is a function of the transverse contact ratio  $\epsilon_\alpha$ . For spur gears it decreases the calculated stress from the AGMA value by 9% when  $\epsilon_\alpha$  is 1.5. There is no equivalent factor for spur gears in the AGMA standards.  $Z_\epsilon$  is also used to interpolate between spur gears where  $\epsilon_\beta$  is zero and true helical gears where  $\epsilon_\beta$  is 1.0 or more.

- The value of  $Z_\beta$  is  $\sqrt{\cos\beta}$ . With  $Z_\epsilon$  it approximates the value of  $F/l_{\min}$  in the AGMA standards. The value tends to "run away" at high helix angles, so ISO

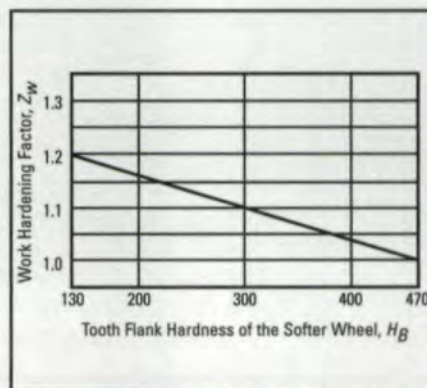


Fig. 2 — Work hardening factor,  $Z_W$ . Based on ISO 6336-2.

6336-1 suggests that users confirm their results by experience when operating helix angles exceed  $30^\circ$ . It is equal to 1.0 for spur gears.

The additional terms in the ISO equations are:

- $K_A, K_v$ , etc. are the general influence factors from ISO 6336-1.
- $S_{H \min}$  is the minimum safety factor input for contact stress.
- $Z_{NT}$  is the life factor from Figure 1.

Note that the life factors are similar to but different from AGMA practice, with a higher value if some pitting is permitted.

•  $Z_L, Z_v$  and  $Z_R$  are related influence factors accounting for the effects of lubricant viscosity, surface roughness and pitch line velocity on the permissible contact stress. Additional inputs for surface finish and lubricant viscosity are required. The combined effect of these three factors is usually less than 10% for industrial gears.

•  $Z_W$  is similar to the AGMA work hardening factor and accounts for the beneficial effect of running a harder pinion against a softer gear. Values are taken from Figure 2.

•  $Z_X$  is a size factor used to reduce the permissible stress for coarse pitch gears. It accounts for the greater possibility of encountering a material defect in the larger stressed volume of larger gears. The size factor is set to 1.0 for contact stress.

#### Root bending stresses

ISO 6336-3 shows how to calculate the root fillet tensile stress and the permissible bending stress. It is based on cantilever beam stress theory similar to AGMA 2001, with several significant differences:

• The point of critical stress is taken at the point on the root fillet which is tangent to an inscribed equilateral triangle, rather than the Lewis parabola. The location of this critical stress point agrees well with the other standards for "normal" gears, but diverges for gears with a small number of teeth and for gears with high operating pressure angles. ISO/TC60/SC2/WG6 has appointed an ad hoc group to study this area of the standard, but results are expected to take a few years. ISO 6336-1 suggests that users confirm their results by experience when operating pressure angles exceed  $25^\circ$  (Fig. 3).

• The stress concentration factors are based on strain gauge research performed on test gears at FZG<sup>1</sup>, rather than on the photo elastic research of Dolan and Broghammer.

• The effect of the compressive component of tooth loading on the root tensile stress is ignored.

• There is no rim thickness factor similar to AGMA's  $K_B$ .

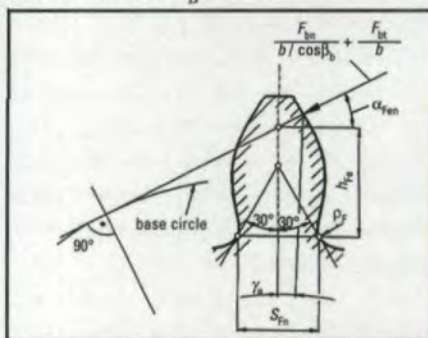


Fig. 3 — Determination of the normal chordal dimensions of the tooth root critical section for Method B. Based on ISO 6336-3.



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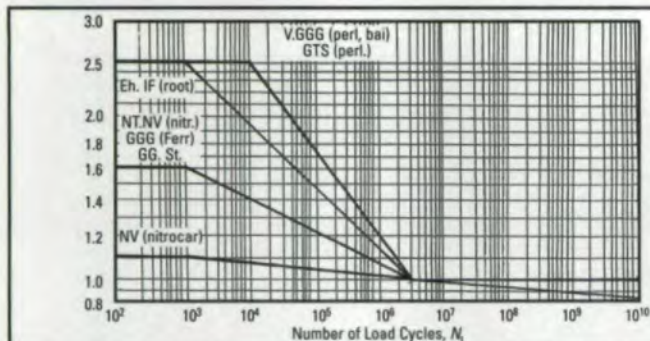


Fig. 4 — Life factor,  $Y_{NT}$ , for standard reference test gears. Based on ISO 6336-3.

The fundamental ISO root bending stress equations are:

$$\sigma_F = \sigma_{FO} K_A K_V K_{FB} K_{F\alpha} \leq \sigma_{FP} \quad \text{ISO6336-3...(1)}$$

$$\sigma_{FO-B} = \frac{F_t}{b m_n} Y_F Y_S Y_\beta \quad \text{ISO6336-3...(2)}$$

$$\sigma_{FP} = \frac{\sigma_{H \text{ lim}} Y_{ST} Y_{NT}}{S_{F \text{ min}}} Y_{\sigma \text{ rel } T} Y_{P \text{ rel } T} Y_X \quad \text{ISO6336-3...(3)}$$

where:

$\sigma_F$  is the calculated bending stress, similar to  $s_t$  in AGMA 2001.

$\sigma_{FO-B}$  is the nominal bending stress. The “-B” in the subscript indicates that method B is being used. An optional simplified method to find the geometry factors from graphs is called method C.

$\sigma_{FP}$  is the permissible bending stress.

$\sigma_{F \text{ lim}}$  is the allowable bending stress number from ISO 6336-5, similar to  $s_{at}$  in AGMA 2001. The terms  $Y_F$ ,  $Y_S$  and  $Y_\beta$  serve the same purpose as the AGMA  $J$  factor. The values are not the same because of the differences in underlying assumptions described above.

The additional terms in the ISO equations are:

- $S_{F \text{ min}}$  is the minimum safety factor input for bending stress.
- $Y_{ST}$  is an experimental stress correction factor, set to 2.0.
- $Y_{NT}$  is the life factor from Figure 4. Note that the life factors are similar to but different from AGMA practice.

•  $Y_{\sigma \text{ rel } T}$  and  $Y_{P \text{ rel } T}$  are material related influence factors which account for the effects of notch sensitivity and surface roughness on the permissible bending stress.

•  $Y_X$  is a size factor used to reduce the permissible stress for coarse pitch gears. It accounts for the greater possibility of encountering a material defect in the larger stressed volume of larger gears. The size factor is taken from Figure 5.

#### Legend

St	= Steel ( $\sigma_B < 800 \text{ N/mm}^2$ )
V	= Through-hardening steel, through-hardened ( $\sigma_B \geq 800 \text{ N/mm}^2$ )
GG	= Grey cast iron
GGG (perl, bai, ferr.)	= Nodular cast iron (perlitic, bainitic, ferritic structure)
GTS (perl.)	= Black malleable cast iron (perlitic structure)
Eh	= Case-hardening steel, case hardened
IF	= Steel and GGG, flame or induction hardened
NT (nitr.)	= Nitriding steel, nitrided
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## CORRECTION

Errors were introduced during editing to Fig. 3 of "Comparing Standards: The keys to understanding ISO 6336-1 gear rating" in the September/October 1998 issue. Below is the corrected illustration.

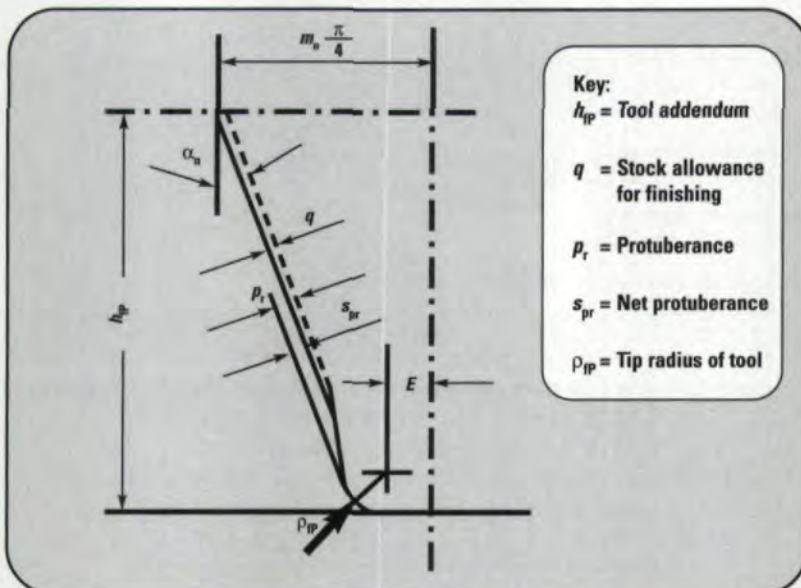
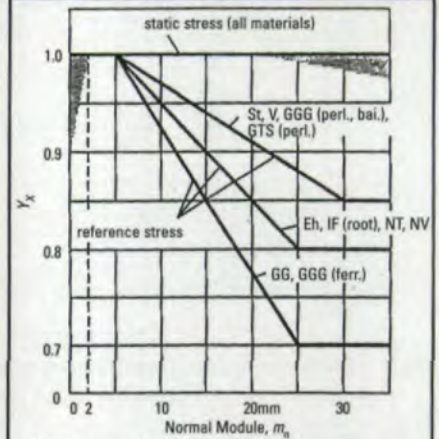


Fig. 3 — Dimensions and basic rack profile of the teeth (finished profile with undercut).

ISO 6336



\*Shaded area is the range of scatter for static stress.

Legend	
St	= Steel ( $\sigma_b < 800 \text{ N/mm}^2$ )
V	= Through-hardening steel, through-hardened ( $\sigma_b \geq 800 \text{ N/mm}^2$ )
GG	= Grey cast iron
GGG (perl., bai., ferr.)	= Nodular cast iron (perlitic, bainitic, ferritic structure)
GTS (perl.)	= Black malleable cast iron (perlitic structure)
Eh	= Case-hardening steel, case hardened
IF	= Steel and GGG, flame or induction hardened
NT (nitr.)	= Nitriding steel, nitrided
NV (nitr.)	= Through-hardening and case-hardening steel, nitrided
NV (nitrocar.)	= Through-hardening and case-hardening steel, nitrocarburized

Fig. 5 — Size factor,  $Y_x$ , for tooth bending strength. Based on ISO 6336-3.

The material properties to be used in these stress equations come from ISO 6336-5, *Strength and quality of materials*. The ISO material quality requirements and allowable stress levels are similar to AGMA's. The differences in material requirements will be pointed out in the last article of this series.

It is difficult to make a general comparison between the ISO and AGMA gear capacity calculation methods, since the details of the individual example can have a big effect on the results. We'll demonstrate this by comparing the calculated capacities of some actual gear sets by the ISO and AGMA methods in that article. ☉

### References:

- Hirt, M. *Einfluß der Zahnfußabrundung und des Festigkeit von Geradstirnrädern*. Doctorate dissertation, Technische Universität München, 1974. Also available in English translation from AGMA.
- Brossmann, U. *Über den Einfluß der Zahnfußabrundung und des Schrägungswinkels auf Beanspruchung und Festigkeit schrägverzählter Stirnräder*. Doctorate dissertation, Technische Universität München, 1979.

### Don McVittie

is one of Gear Technology's technical editors. He is president of Gear Engineers, Inc., Seattle, WA and a former president of AGMA. McVittie is a licensed professional engineer in the state of Washington and has been involved with gear standards development for more than 25 years.

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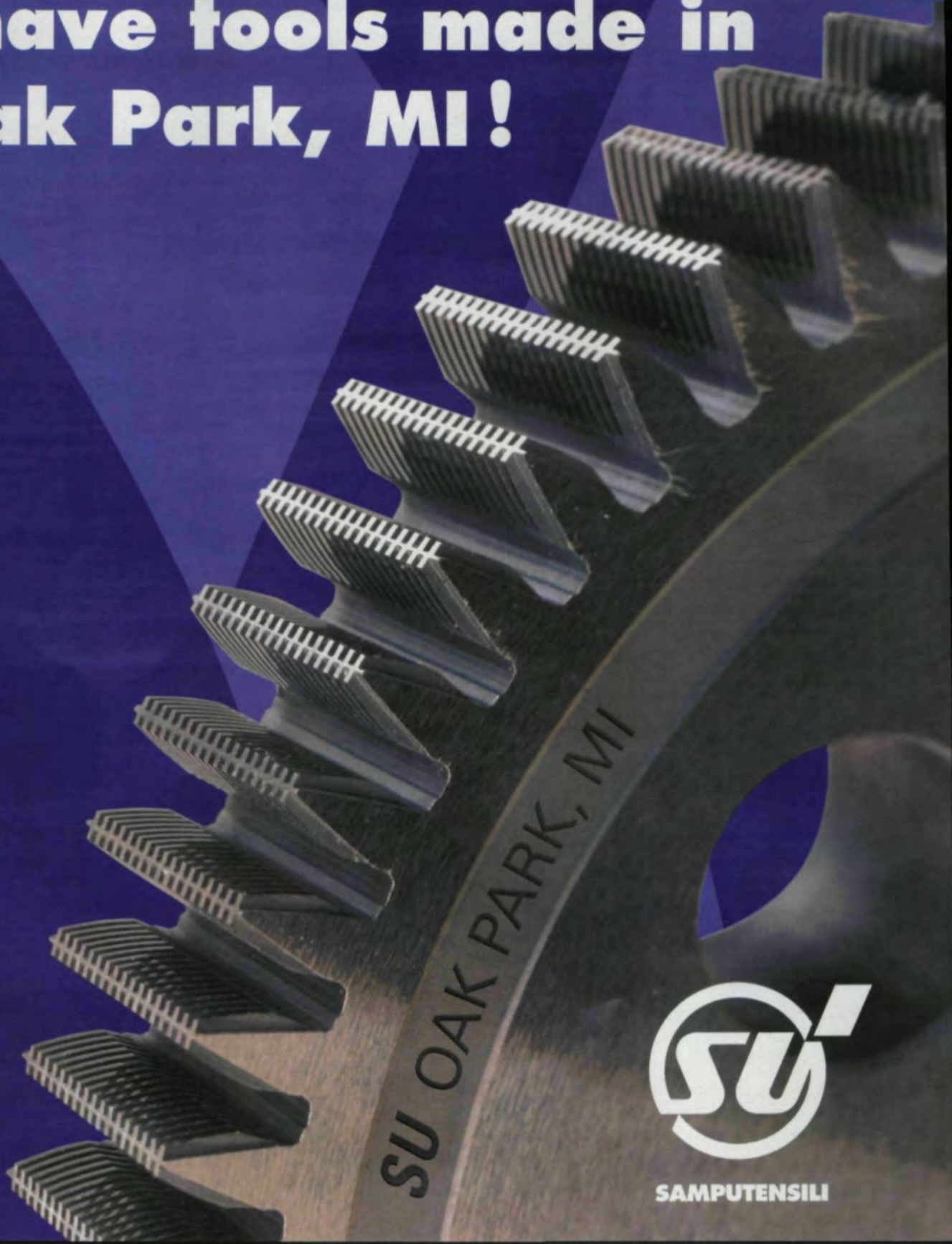
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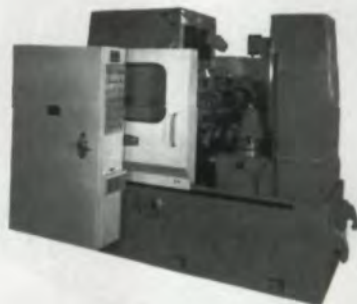
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CIRCLE 109

# Gears on the Firing Line

*High-volume centrifugal compressors push the limits of gear quality and reliability in this case study from Acme Gear, Cooper Turbocompressor and Reishauer*

Dennis Richmond

**A**ir compressors are a good example of industrial machinery with components that rotate at very high speeds, up to 80,000 rpm. They are subject to very high rotational forces and often variable loads. Strong, high-precision gears for the power transmission trains that drive the impellers are critical components of machinery operating under such conditions.

Compressors often operate around the clock in these applications. High reliability (up time), energy efficiency and vibration-free operation (typically, gear noise cannot be heard over the gas compression and flow noise) of compressors are important considerations. The results of a gearbox failure can be catastrophic—resulting in plant operation shut down and damage to drive shafts and impellers. Repairs or rebuilds can be costly.

Cooper Turbocompressor in Buffalo, NY, manufactures a wide range of oil-free centrifugal compressors for plant air and process applications. Typically they are the source of compressed air for process machine tool operation, pneumatic tools and even snow-making equipment, with power ratings from 150 to more than 1,200 horsepower. Cooper is also an industry leader for compressors in the air separation industry (suppliers of liquid oxygen, nitrogen and other gases) up to 22,000 hp.

Acme Gear supplies Cooper Turbocompressor with dynamically balanced, single helical bull gears, in sizes up to 32.68 in. pitch diameter, and pinions for their TA2000, TA3000 and C-8



Fig. 1 – Joe Gelles, president of Acme gear, with the modified Reishauer RZ820 used to grind centrifugal compressor gears.

centrifugal compressor lines as well as custom-engineered compressors rated between 2,000 and 7,000 hp. In many instances Acme supplies all of the gearing for a particular compressor. Acme also provides gearing for custom-engineered compressors, some with two separate gearboxes run in series off the same drive motor. Reishauer CNC gear grinding machines are employed for production, including a new Reishauer RZ820 for the larger bull gears.

The gearboxes and drive trains of Cooper's single and multi-stage compressors are an integral part of the entire assembly. A single main drive or bull gear will drive one to three pinions. They must reliably drive compressor impellers to pinion speeds of up to 80,000 rpm, provide vibration-free rotation of all drive shafts and impellers and balance out and absorb varying thrust loads for high-efficiency operation.

Centrifugal compressor gearing is some of the most difficult to produce due



Fig. 2 – Cooper Turbocompressor gear being ground with 5° bevel.



Fig. 3 – Finished bull gear being removed from Reishauer RZ820.

to its complexity and the stringent requirements for material quality and dimensional tolerances. And, consistent, high-precision machining of hard alloy steels is a tough job that places great demands on both operations and the production machinery used to make them.

Acme's compressor gears are manufactured with through-hardened, aircraft-quality alloy steels, which are subjected to both destructive and nondestructive

testing procedures. Tolerances of 1/10,000 (.0001) in. are necessary. Some specific areas require tolerances held to +/- fifty millionths (.00005) in. In the case of ground, hardened steel parts, there is no distortion due to machining stresses. Final tolerances are achieved without subsequent stress relief and grinding or polishing, as might be the case with gears produced by gear hobbing or shaping machines.

The gearing Acme supplies is ground to AGMA 12 and 13+ levels. Cooper also has its own set of proprietary specifications related to the manufacturing of centrifugal compressor quality gearing, and Cooper's operations are ISO 9001 certified.

Cooper Turbocompressor's quality department audits Acme three to four times a year in addition to an annual "Supplier Quality Performance Review," which is a thorough review of products and procedures on both a quantitative and qualitative basis. Cooper also has very formal rejection and disposition procedures as well as cause and corrective action reporting requirements. Acme Gear consistently scores in the highest 10 percent of their supplier base.

Acme consistently meets such stringent requirements through a close relationship based upon mutual benefit and respect both with Cooper and with their own suppliers of materials and production machinery. For Cooper, it is very important that suppliers employ, maintain and upgrade their machinery and processes to the levels necessary to reliably and consistently produce the gearing quality that Cooper and, ultimately, their customers require.

#### Supplying the High Performance Gears

Originally founded as Marine Boat Engine Company in 1929, Acme Gear of Englewood, NJ, was initially in the business of rebuilding marine engines. In 1955, the company moved from Long Island City to its present site in Englewood and was renamed Acme, "because it represents the highest point,



Fig. 4 - Each turbocompressor gear is quality checked for accuracy and dimensionality of bevels.

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the culmination," states Joe Gelles, Acme president.

In the mid-1960s, the business began to shift. Acme became heavy in the cut gear business. Government contractors such as Grumman, Fairchild and Republic were relocating, cutting back or just closing down. Around 1967, Acme decided to make the shift from cut gears to precision ground gears because of the evolving niche requiring high quality components.

Acme Gear began to produce high speed, precision gears for existing customers like Borg Warner Corporation, Carrier Industries, Bendix and Hillcraft. Demands on the gearing have changed dramatically in the last 30 years. In 1967, "high speed" was around 2,000 rpm. Today, Acme produces precision gears capable of about 120,000 rpm. "It's more than just speed," says Gelles. "In the printing industry, for example, magazine presses have more than seven colors being overlapped while running at tremendous speeds. The precision required to maintain those exacting registrations demands a lot from the gearing."

Acme Gear has recently installed a modified version of a Reishauer RZ820 machine. It enables Acme to produce a 32.68" precision gear for Cooper Cameron as part of a high efficiency refrigeration compressor. Acme worked in conjunction with Reishauer to modify an RZ820 to meet the customer's needs for a larger workpiece from the standard 32" capacity. "Acme had been producing a lot of gearing in the 27" range, which fit wonderfully on the other Reishauer RZ series machines, but when a new, larger design came up at Cooper, we sat down with the Reishauer people and together came up with the perfect solution," says Gelles.

Other new features on the RZ820 machine in addition to the larger capacity included a more powerful grinding motor (5.5 kw), variable speed grinding spindle (1100-2150 rpm) that can be selected for individual passes, manually actuated fine balancing and hydraulic tailstock.

Joe Gelles predicts an increase in the usage of high speed precision gears throughout the industry. The demand for

more efficient performance in compressors, elevators, mining equipment and locomotives. As for the automotive industry, Gelles concludes, "While they're shaving and honing a lot of them still, the consumer will demand transmissions that will last longer, make less noise and operate more efficiently. This will be especially true as the auto industry moves into the era of electric cars. We'll be ready for them when it happens." ☉

**Dennis Richmond**

*is vice president of Reishauer Corporation, Elgin, IL, manufacturers of precision gear grinding machinery.*

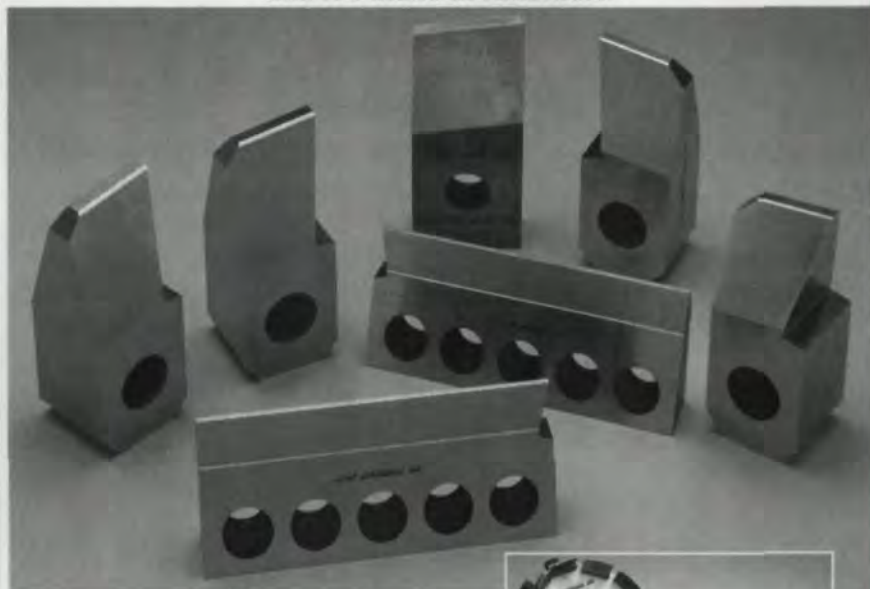
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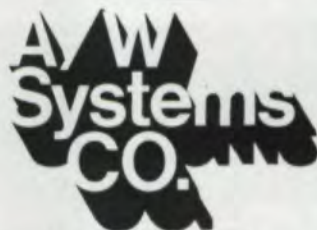
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Precipart Corp.  
Precision Gear Inc.  
**The Purdy Corporation**  
R. Cushman & Assoc.  
R.R. Transmissions  
Rapid Gear  
Reliance Gear Co. Ltd.  
Reliance Gear Corp.  
Renold Power Transmission Corp.  
Rexnord Corporation  
Rj Link International  
Rockwell Automation/Dodge

Ronson Gears Pty. Ltd.  
Rotodrives  
Santasalo North America  
Satellite Gear  
Seitz Corporation  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Snow-Nabstedt Power Transmission  
Springer Company Inc.  
Sri Venkateshwar Gear  
Standard Industrial Products Co.  
STD Precision Gear & Instrument  
Sterling Electric  
Stober Drives  
Stock Drive Products/  
Sterling Instrument  
Sumitomo Machinery  
Sussex Gear Company  
Teijin Seiki/NIMAC America  
Thor Technology  
Torque Transmission  
Tri-Power MPT  
Trogetec Inc.  
United States Gear  
V.T.M. Co. Ltd.  
Von Ruden Mfg.  
Walter Machine Co., Inc.  
Wedin International  
Westech Gear  
Westerman Companies  
Wes-Tex Gear Inc.  
Weyers Brothers (FT) Ltd.  
Zenith Sintered Products  
Zero-Max, Inc.  
ZF Industries

### Gear Couplings

Acme Gear Co., Inc.  
ACR Industries, Inc.  
The Adams Company  
Amarillo Gear Co.—Russellville  
American Metric Corp.  
Bearings & Industrial Sales  
Boston Gear  
Brewer Machine & Gear  
Cloyes Gear & Products—Auburn Hills  
Davall Gear Co. Ltd.  
Dayton Gear & Tool  
Dee-Kay Gears  
Elmass North America  
Foote-Jones/Illinois Gear  
Gear Company of America  
Gears & Drive Systems  
Great Taiwan Gear Ltd.  
Harder Precision Components  
HCI Supply  
HPC Drives Ltd.  
Industrial Supply Co.  
Invincible Gear Co.  
J&E Hofmann Engineering  
Link Gear & Machine  
Lovejoy, Inc.  
Milwaukee Gear Co.  
Minipart P.T. Co. Ltd.  
Nordex, Inc.  
Orlandi Gear  
Overtone Gear & Tool

Pennsylvania Gear Corp.  
**Perry Technology Corp.**  
PIC Design  
Precision Gear Inc.  
Precision Gears, Inc.  
Pulley Manufacturers Inc.  
R.R. Transmissions  
Rj Link International  
Ronson Gears Pty. Ltd.  
Rush Gears Inc.  
Santasalo North America  
Satellite Gear  
Standard Industrial Products Co.  
Stock Drive Products/  
Sterling Instrument  
Trojan Gear Inc.  
Wedin International

### Gear Motors

American Metric Corp.  
Baldor Motors & Drives  
Bauer Gear Motors  
Bearings & Industrial Sales  
Bison Gear & Engineering  
Bonfiglioli Riduttori  
Brad Foote Gear Works  
Brook Hansen  
Caron-Vector  
C-B Gear & Machine  
Comer Group  
David Brown Group PLC  
Electra-Gear  
Euclid Universal Corp.  
Fleet Tools Ltd.  
Foote-Jones/Illinois Gear  
Gears & Drive Systems  
Greenspon Engineering Works Ltd.  
Groschopp  
Grove Gear  
Harmonic Drive Technologies  
HPC Drives Ltd.  
Hub City, Inc.  
Indiana Power Transmission Systems  
Industrial Supply Co.  
ITW Spiroid  
Koellmann Gear  
Maquinaria ESBO  
Minipart P.T. Co. Ltd.  
Nissei Corp. of America  
Nord Gear Corporation  
Ohio Gear  
Omni Gear  
Parvalux USA  
Renold Power Transmission Corp.  
Rexnord Corporation  
Rotodrives  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Sri Venkateshwar Gear  
Standard Industrial Products Co.  
Stober Drives  
Stock Drive Products/  
Sterling Instrument  
Teijin Seiki/NIMAC America  
Thor Technology  
Tri-Power MPT  
Wedin International

Weyers Brothers (FT) Ltd.

### Gears—Face

ACR Industries, Inc.  
Action Gear & Broaching  
Adobe Precision Gear  
Akron Gear & Engineering  
American Precision Gear  
Bengal Industries  
C-B Gear & Machine  
Cloyes Gear & Products—Auburn Hills  
Davall Gear Co. Ltd.  
Dayton Gear & Tool  
Elmass North America  
Fisher's Gear & Machine  
Foote-Jones/Illinois Gear  
G&N Rubicon Gear  
Gears & Drive Systems  
Generated Gear & Machine  
Great Taiwan Gear Ltd.  
Harder Precision Components  
Intech Corporation  
J&E Hofmann Engineering  
M.G. Minigears Inc.  
**National Broach & Machine Co.**  
Nordex, Inc.  
Oliver Gear, Inc.  
**Perry Technology Corp.**  
Precision Gear Inc.  
Pulley Manufacturers Inc.  
Ronson Gears Pty. Ltd.  
Sales Consultants  
Santasalo North America  
Schafer Gear Works, Inc.  
STD Precision Gear & Instrument  
Stock Drive Products/  
Sterling Instrument  
UFE Incorporated  
Wedin International

### Gears—Helical

A-1 Gears  
ABA-PGT Inc.  
Acme Gear Co., Inc.  
ACR Industries, Inc.  
Action Gear & Broaching  
The Adams Company  
Adobe Precision Gear  
Advance Gear & Machine  
Aero Gear  
Aerocom Industries Inc.  
Akron Gear & Engineering  
Allied Devices Corp.  
**Allied Gear Co.**  
**Amarillo Gear Co.—Amarillo**  
Amarillo Gear Co.—Russellville  
American Gear & Engineering  
American Mach. & Gear  
American Metric Corp.  
American Precision Gear  
Aplus Engineering Inc.  
Arrow Gear Company  
Ashot Ashkelon Indust.  
Ashot USA Inc.  
**Axicon Technologies**

Bearings & Industrial Sales  
Bengal Industries  
Bevel Gears (India)  
Bilgram Gear Co.  
Bonfiglioli Riduttori  
Bonfiglioli U.K.  
Boonville Mining Services  
Boston Gear  
Boxx Gear Mfg., Inc.  
Brad Foote Gear Works  
Brewer Machine & Gear  
Buckeye Gear Co.  
Burgess-Norton Mfg. Co.  
CMD (UK) Ltd.  
Calicut Eng. Works Ltd.  
Capstan Atlantic  
Carbon City Products  
Caterpillar Industrial Products Inc.  
C-B Gear & Machine  
Chardam Gear Co.  
Charles Bond Co.  
Chicago Gear-D. O. James  
Chicago Gear Works  
Ciateq, A. C.  
The Cincinnati Gear Co.  
Circle Gear & Machine  
Cloyes Gear & Products—Auburn Hills  
Columbia Gear Corp.  
Commercial Gear & Sprocket  
**Crown Gear B.V.**  
Curtis Machine Co. Inc.  
Custom Gear & Machine  
Custom Gears, Inc.  
Dabko Industries Inc.  
Davall Gear Co. Ltd.  
David Brown Group PLC  
Dayton Gear & Tool  
Dearborn Gear & Tool Co.  
Dee-Kay Gears  
E.C. Machining, Inc.  
Electrex Ltd. (India)  
Elmass North America  
EMCO Gears, Inc.  
Engelhardt Gear Co.  
Euclid Universal Corp.  
Fairfield Mfg. Co.  
Falk Corp.  
Federal Gear Corp.  
Fisher's Gear & Machine  
Fleet Tools Ltd.  
Flender Corporation  
Foote-Jones/Illinois Gear  
Franke Gear Works Inc.  
G&N Rubicon Gear  
GA-Heartland Machine Tool  
Gear Company of America  
Gear Systems Inc.  
Gear Works, Inc.  
The Gear Works-Seattle  
Gears & Drive Systems  
Geartronic Industries  
General Gear Corporation  
Generated Gear & Machine  
Gerhardt Gear Co.  
Global Gear  
Globe Gear Co.  
Great Taiwan Gear Ltd.  
Greenspon Engineering Works Ltd.

GW Plastics  
Hamer Gear  
Hamilton Gear  
Hanover Gear Mfg. Co.  
Harder Precision Components  
HCI Supply  
Hico  
Hitachi America Ltd.  
HMC  
**Holroyd Machine**  
Horsburgh & Scott  
Hub City, Inc.  
Indiana Power  
Transmission Systems  
Industrial Supply Co.  
Inscop Corporation  
Intech Corporation  
Invincible Gear Co.  
Invo Spline Inc.  
Involute Tooling Corp.  
J&E Hofmann Engineering  
Jackson Gear Co.  
Jade Precision Gear Co.  
Jennings Machine & Gear  
KA-Wood Gear  
Keller Machine Co.  
Koellmann Gear  
Kreiter Geartech  
Krupp Engineering Inc.  
L&H Welding & Machine  
Lampin Corp.  
Lawler Gear Corp.  
Link Gear & Machine  
Linn Gear Co.  
Lufkin Industries Gear Repair  
Lyon Gear  
M.G. Minigears Inc.  
M.J.H. Gear & Tool Co.  
Mascotech-Braun  
Merit Gear Corp.  
Micron Instrument Corp.  
**Midwest Gear Corp.**  
**Midwest Gear & Tool**  
Milford Gear Works  
Milwaukee Gear Co.  
Molon Gear & Shaft  
Moore Gear Mfg. Co.  
Moore Machine & Gear  
Mostar Gear & Machine  
Mr. Gears, Inc.  
Murray Brothers Mfg. Co.  
**Nakanishi Gear**  
**National Broach & Machine Co.**  
**Niagara Gear Corp.**  
Nissei Corp. of America  
Nixon Gear  
Nord Gear Corporation  
Norflex, Inc.  
Nuttall Gear Corp.  
OEM Industries Inc.  
Oliver Gear, Inc.  
Omni Gear  
Omni Gear & Machine  
O'Neill Gear  
**Ontario Drive & Gear**  
Overton Gear & Tool Corp.  
P.T. International Corp.  
Patterson Gear & Machine  
Penn Machine Company  
Pennsylvania Gear Corp.  
Penntech  
**Perry Technology Corp.**  
Philadelphia Gear Corp.—  
King of Prussia  
Philadelphia Gear Corp.—  
Houston  
PIC Design  
Poly Hi Solidur  
Power Engineering & Mfg. Ltd.  
Precipart Corp.  
Precision Gear Co.  
Precision Gear Inc.  
Precision Gears, Inc.

**Presrite Corp.**  
Process Industries  
Production Gear & Broach  
Progressive Engineering  
**The Purdy Corporation**  
Qualicast Corp.  
**Quality Transmission Components**  
R.L. Wagner & Assoc.  
Rapid Gear  
Rawling Gear Inc.  
Reef Gear Mfg. Inc.  
Reliance Gear Co. Ltd.  
Reliance Gear Corp.  
Riley Gear Corp.  
Riverside Spline & Gear  
Rj Link International  
Ronjon Gears Pty. Ltd.  
Rush Gears Inc.  
Sales Consultants  
Santasalo North America  
Satellite Gear  
Schafer Gear Works, Inc.  
Seitz Corporation  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Springer Company Inc.  
Sri Venkateshwara Gear  
Standard Industrial Products Co.  
STD Precision Gear & Instrument  
Sterling Electric  
Stock Drive Products/  
Sterling Instrument  
Suda International Gear Works  
Tech Sales Inc.  
Textile Parts & Machine  
**Tifco Gage & Gear**  
Tracey Gear & Machine  
Tri-Power MPT  
Trojan Gear Inc.  
UFE Incorporated  
United States Gear  
V.T.M. Co. Ltd.  
Von Ruden Mfg.  
Walter Machine Co., Inc.  
Wedin International  
West Industries Inc.  
Westech Gear  
**Western Spline Gage**  
Wes-Tex Gear Inc.  
Wohler Corporation  
Worcester Gear Works  
Worrall Grinding Co.  
Xtek Inc.  
**Gears—Herringbone**  
ACR Industries, Inc.  
Action Gear & Broaching  
Adobe Precision Gear  
Akron Gear & Engineering  
American Mach. & Gear  
Anderson International  
Aplus Engineering Inc.  
Bearings & Industrial Sales  
Bilgram Gear Co.  
Boonville Mining Services  
Brad Foote Gear Works  
CMD (UK) Ltd.  
Calicut Eng. Works Ltd.  
C-B Gear & Machine  
Chicago Gear-D. O. James  
Ciateq, A. C.  
The Cincinnati Gear Co.  
Circle Gear & Machine  
Custom Gear & Machine  
Custom Gears, Inc.  
David Brown Group PLC  
Engelhardt Gear Co.  
Falk Corp.  
Federal Gear Corp.  
Foote-Jones/Illinois Gear  
The Gear Works-Seattle

Gears & Drive Systems  
Globe Gear Co.  
Great Taiwan Gear Ltd.  
Hamer Gear  
HMC  
Horsburgh & Scott  
Industrial Supply Co.  
Intech Corporation  
J&E Hofmann Engineering  
Kreiter Geartech  
L&H Welding & Machine  
Linn Gear Co.  
Lufkin Industries Gear Repair  
**National Broach & Machine Co.**  
Penn Machine Company  
**Perry Technology Corp.**  
Philadelphia Gear Corp.—  
Houston  
Precision Gear Inc.  
Process Industries  
Progressive Engineering  
Qualicast Corp.  
**Quality Transmission Components**  
Rapid Gear  
Rush Gears Inc.  
Sales Consultants  
Santasalo North America  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Springer Company Inc.  
Sri Venkateshwara Gear  
Standard Industrial Products Co.  
Westech Gear  
Western Companies  
Wes-Tex Gear Inc.  
Xtek Inc.  
**Gears—Internal**  
A-1 Gears  
Aerocom Industries Inc.  
Allied Devices Corp.  
**Allied Gear Co.**  
American Mach. & Gear  
Aplus Engineering Inc.  
Avon Bearings  
Bevel Gears (India)  
Bilgram Gear Co.  
Blanchat Machine Co.  
Brad Foote Gear Works  
Brewer Machine & Gear  
CMD (UK) Ltd.  
Calicut Eng. Works Ltd.  
Carbon City Products  
Caterpillar Industrial Products Inc.  
Chandler Machine Co.  
Chardam Gear Co.  
Charles Bond Co.  
Chicago Gear-D. O. James  
Chicago Gear Works  
The Cincinnati Gear Co.  
Commercial Gear & Sprocket  
Custom Gear & Machine  
Custom Gears, Inc.  
David Brown Group PLC  
Dee-Kay Gears  
Dynamic Tool Grinding  
Engelhardt Gear Co.  
Fairfield Mfg. Co.  
Federal Gear Corp.  
Fleet Tools Ltd.  
Franke Gear Works Inc.  
The Gear Works-Seattle  
Geartronic Industries  
Gerhardt Gear Co.  
Globe Gear Co.  
Greenshpon Engineering Works Ltd.  
HMC  
Hamilton Gear  
Hanover Gear Mfg. Co.  
Horsburgh & Scott  
Indiana Tool/Indiana Gear  
Industrial Supply Co.  
Invincible Gear Co.

Jade Precision Gear Co.  
KA-Wood Gear & Machine  
Krupp Engineering Inc.  
Link Gear & Machine  
Linn Gear Co.  
Lyon Gear  
M.J.H. Gear & Tool Co.  
Mascotech-Braun  
Micron Instrument Corp.  
**Midwest Gear & Tool**  
Milford Gear Works  
Modified Gear & Spline  
Moore Machine & Gear  
Mr. Gears, Inc.  
**Nakanishi Gear**  
Nissei Corp. of America  
Nixon Gear  
Omni Gear & Machine  
O'Neill Gear  
**Ontario Drive & Gear**  
Patterson Gear & Machine  
Penn Machine Company  
Penntech  
Philadelphia Gear Corp.—  
Houston  
Philadelphia Gear Corp.—  
King of Prussia  
Precipart Corp.  
Precision Gear Co.  
Process Industries  
Production Gear & Broach  
Progressive Engineering  
Rapid Gear  
Reef Gear Mfg. Inc.  
Reliance Gear Co. Ltd.  
Reliance Gear Corp.  
Riley Gear Corp.  
Riverside Spline & Gear  
Seitz Corporation  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Springer Company Inc.  
Sri Venkateshwara Gear  
STD Precision Gear  
Textile Parts & Machine  
**Tifco Gage & Gear**  
Trogetec Inc.  
United States Gear  
Von Ruden Mfg.  
West Industries Inc.  
Westech Gear  
Winzler Gear  
Worcester Gear Works  
Xtek Inc.  
**Gears—Non-Circular**  
ACR Industries, Inc.  
Action Gear & Broaching  
Advance Gear & Machine  
Akron Gear & Engineering  
Bilgram Gear Co.  
Calicut Eng. Works Ltd.  
Cunningham Industries  
Dallav Gear Co. Ltd.  
Gear Company of America  
Gear Works, Inc.  
Globe Gear Co.  
Great Taiwan Gear Ltd.  
GW Plastics  
Horsburgh & Scott  
Krupp Engineering Inc.  
Merit Gear Corp.  
**Midwest Gear & Tool**  
**Perry Technology Corp.**  
Stock Drive Products/  
Sterling Instrument  
Trogetec Inc.  
Waterjet Connection/  
Richel Inc.  
Wedin International  
Worcester Gear Works  
**Gears—Planetary**  
Bearings & Industrial Sales  
Brad Foote Gear Works

Brewer Machine & Gear  
CMD (UK) Ltd.  
Capitol Stampings Corp.  
David Brown Group PLC  
Dee-Kay Gears  
Gerhardt Gear Co.  
Greenshpon Engineering Works Ltd.  
Invincible Gear Co.  
Masiero Antonio SpA  
Minipart P.T. Co. Ltd.  
Mr. Gears, Inc.  
Rapid Gear  
Seitz Corporation  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Snow-Nabstedt Power Transmission  
Springer Company Inc.  
Standard Industrial Products Co.  
STD Precision Gear  
**Gears—Plastic**  
ABA-PGT Inc.  
Allied Devices Corp.  
American Mach. & Gear  
Arrow Gear Company  
Bearings & Industrial Sales  
Bengal Industries  
Bilgram Gear Co.  
Brewer Machine & Gear  
Calicut Eng. Works Ltd.  
Capitol Stampings Corp.  
Chicago Gear-D. O. James  
Chicago Gear Works  
Commercial Gear & Sprocket  
Dabko Industries Inc.  
Dallav Gear Co. Ltd.  
David Brown Group PLC  
Fisher's Gear & Machine  
GW Plastics  
Gear Systems Inc.  
The Gear Works-Seattle  
Gears & Drive Systems  
Geartronic Industries  
Globe Gear Co.  
Great Taiwan Gear Ltd.  
HCI Supply  
Industrial Supply Co.  
ITW Spiroid  
Jennings Machine & Gear  
KA-Wood Gear & Machine  
M.J.H. Gear & Tool Co.  
Martin Sprocket & Gear  
**Midwest Gear & Tool**  
Moore Machine & Gear  
Mr. Gears, Inc.  
Omni Gear & Machine  
Performance Gear Systems Inc.  
Philadelphia Gear Corp.—  
Houston  
Precision Gear Co.  
Process Industries  
Progressive Engineering  
Putnam Precision Molding  
**Quality Transmission Components**  
R.L. Wagner & Assoc.  
Rapid Gear  
Reliance Gear Corp.  
Riley Gear Corp.  
Rush Gears Inc.  
Seitz Corporation  
Spicer Industries  
Standard Industrial Products Co.  
Stock Drive Products/  
Sterling Instrument  
Textile Parts & Machine  
**Tifco Gage & Gear**  
Transmission Developments Co. Ltd.  
Trogetec Inc.

UFE Incorporated  
Wedin International  
Winzler Gear  
Worcester Gear Works  
**Gears—Powder Metal**  
Aplus Engineering Inc.  
**Asco Sintering Co.**  
Ashot USA Inc.  
Bearings & Industrial Sales  
Bestmetal Corporation  
Burgess-Norton Mfg. Co.  
Capstan Atlantic  
Carbon City Products  
Cloyes Gear & Products—  
Auburn Hills  
Dabko Industries Inc.  
David Brown Group PLC  
Dee-Kay Gears  
Euclid Universal Corp.  
Foote-Jones/Illinois Gear  
The Gear Works-Seattle  
Great Taiwan Gear Ltd.  
HCI Supply  
Industrial Supply Co.  
ITW Spiroid  
Keystone Powdered Metal  
Lyon Gear  
M.G. Minigears Inc.  
Martin Sprocket & Gear  
Masiero Antonio SpA  
Metal Powder Components  
Minipart P.T. Co. Ltd.  
Pennsylvania Pressed Metal Inc.  
Precision Gear Co.  
St. Marys Carbon Co.  
Standard Industrial Products Co.  
**Tifco Gage & Gear**  
Trogetec Inc.  
Zenith Sintered Products  
**Gears—Racks**  
A-1 Gears  
Akron Gear & Engineering  
Allied Devices Corp.  
American Gear & Engineering  
American Mach. & Gear  
American Metric Corp.  
American Precision Gear  
Anderson International  
Aplus Engineering Inc.  
**Asco Sintering Co.**  
Bearings & Industrial Sales  
Bevel Gears (India)  
Boonville Mining Services  
Boston Gear  
Capstan Atlantic  
Carbon City Products  
C-B Gear & Machine  
Charles Bond Co.  
The Cincinnati Gear Co.  
Circle Gear & Machine  
Commercial Gear & Sprocket  
Cornell Forge Co.  
Custom Gear & Machine  
Dabko Industries Inc.  
Dallav Gear Co. Ltd.  
Dayton Gear & Tool  
Engelhardt Gear Co.  
Federal Gear Corp.  
**Fellows Corp.**  
Fisher's Gear & Machine  
Fleet Tools Ltd.  
Foote-Jones/Illinois Gear  
Franke Gear Works Inc.  
Gear Company of America  
The Gear Works-Seattle  
Gears & Drive Systems  
Geartronic Industries  
Generated Gear & Machine

Globe Gear Co.  
Great Taiwan Gear Ltd.  
GW Plastics  
HMC  
Intech Corporation  
J&E Hofmann Engineering  
Jade Precision Gear Co.  
Keystone Powdered Metal  
Krupp Engineering Inc.  
Lawler Gear Corp.  
Martin Sprocket & Gear  
Modified Gear & Spline  
Moore Gear Mfg. Co.  
Moore Machine & Gear  
Mostar Gear & Machine  
**National Broach & Machine Co.**  
Nixon Gear  
Nordex, Inc.  
Oliver Gear, Inc.  
Patterson Gear & Machine  
**Perry Technology Corp.**  
PIC Design  
Pitch Templates, Inc.  
Ply-Mar Tool Co.  
Poly Hi Solidur  
Precipart Corp.  
Progressive Engineering  
Qualicast Corp.  
**Quality Transmission Components**  
Rack & Pinion, Inc.  
Rapid Gear  
Reliance Gear Co. Ltd.  
Riley Gear Corp.  
Riverside Spline & Gear  
Ronson Gears Pty. Ltd.  
Rush Gears Inc.  
Santasalo North America  
Satellite Gear  
Sri Venkateshwara Gear  
Standard Steel Specialty  
STD Precision Gear  
Stock Drive Products/  
Sterling Instrument  
Suda International Gear Works  
Trogetec Inc.  
Trojon Gear Inc.  
UPE Incorporated  
Wedin International  
Worcester Gear Works  
Xtek Inc.  
Zhuhai Intercontinental  
Pulleys Ltd.

**Gears—Segment**

ABA-PGT Inc.  
Acme Gear Co., Inc.  
ACR Industries, Inc.  
Action Gear & Broaching  
Adobe Precision Gear  
Aero Gear  
Akron Gear & Engineering  
Albro Gear & Instrument  
**Allied Gear Co.**  
American Gear & Engineering  
American Mach. & Gear  
American Precision Gear  
Arrow Gear Company  
**Asco Sintering Co.**  
Boonville Mining Services  
Boxx Gear Mfg., Inc.  
C-B Gear & Machine  
Chandler Machine Co.  
Circle Gear & Machine  
Cloyes Gear & Products—  
Auburn Hills  
Columbia Gear Corp.  
Davall Gear Co. Ltd.  
Dayton Gear & Tool  
Engelhardt Gear Co.  
Fisher's Gear & Machine  
Foote-Jones/Illinois Gear  
GA-Heartland Machine  
Tool  
Gear Company of America

Gears & Drive Systems  
Generated Gear & Machine  
Gerhardt Gear Co.  
Great Taiwan Gear Ltd.  
Hamilton Gear  
Harder Precision Components  
Indiana Power  
Transmission Systems  
J&E Hofmann Engineering  
Kreiter Geartech  
Moore Gear Mfg. Co.  
Mostar Gear & Machine  
Nissei Corp. of America  
Nordex, Inc.  
OEM Industries Inc.  
Oliver Gear, Inc.  
Overton Gear & Tool  
Pennsylvania Gear Corp.  
**Perry Technology Corp.**  
PIC Design  
Production Gear & Broach  
**The Purdy Corporation**  
**Quality Transmission Components**  
Rapid Gear  
Rawling Gear Inc.  
Ronson Gears Pty. Ltd.  
Rush Gears Inc.  
Santasalo North America  
Satellite Gear  
Schafer Gear Works, Inc.  
Sri Venkateshwara Gear  
STD Precision Gear  
Stock Drive Products/  
Sterling Instrument  
Tracey Gear & Machine  
Trojon Gear Inc.  
Waterjet Connection/  
Richel Inc.  
Wedin International

**Gears—Spiral Bevel & Hypoid**

ABA-PGT Inc.  
ACR Industries, Inc.  
Advance Gear & Machine  
Aero Gear  
**Amarillo Gear Co.—Amarillo**  
American Metric Corp.  
Arrow Gear Company  
**Asco Sintering Co.**  
Ashot Ashkelon Indust.  
Ashot USA Inc.  
Astron Midwestern Inc.  
**ATA Gears**  
Bearings & Industrial Sales  
Bevel Gears (India)  
Blanchat Machine Co.  
Bonfiglioli Riduttori  
Bonfiglioli U.K.  
Boston Gear  
Brad Foote Gear Works  
Burgess-Norton Mfg. Co.  
Carbon City Products  
Caron-Vector  
Caterpillar Industrial Products Inc.  
C-B Gear & Machine  
Chicago Gear-D. O. James  
Ciateq, A. C.  
The Cincinnati Gear Co.  
CMD (UK) Ltd.  
Curtis Machine Co. Inc.  
Davall Gear Co. Ltd.  
David Brown Group PLC  
Dee-Kay Gears  
Electrex Ltd. (India)  
Elmass North America  
EMCO Gears, Inc.  
Engelhardt Gear Co.  
Fairfield Mfg. Co.  
Falk Corp.  
Foote-Jones/Illinois Gear  
G&N Rubicon Gear

The Gear Works-Seattle  
Gears & Drive Systems  
Globe Gear Co.  
Great Taiwan Gear Ltd.  
Grupos Diferenciales  
Hamilton Gear  
HCI Supply  
Hico  
Hitachi America Ltd.  
Horsburgh & Scott  
Hub City, Inc.  
Indiana Power  
Transmission Systems  
Industrial Supply Co.  
Intech Corporation  
Keystone Powdered Metal  
Klingelberg Söhne GmbH  
Krupp Engineering Inc.  
Lampin Corp.  
Linn Gear Co.  
M.G. Minigears Inc.  
M.S. Engineers  
Masiero Antonio SpA  
Micron Instrument Corp.  
**Midwest Gear & Tool**  
Milford Gear Works  
Moore Gear Mfg. Co.  
Nissei Corp. of America  
Oerlikon Geartec AG  
Ohio Gear  
Omni Gear  
Philadelphia Gear Corp.—  
Houston  
Philadelphia Gear Corp.—  
King of Prussia  
PIC Design  
Precision Gear Co.  
Precision Gear Inc.  
**Presrite Corp.**  
**The Purdy Corporation**  
Qualicast Corp.  
**Quality Transmission Components**  
Reliance Gear Corp.  
Riverside Spline & Gear  
Rush Gears Inc.  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Springer Company Inc.  
Standard Industrial Products Co.  
Stock Drive Products/  
Sterling Instrument  
Suda International Gear Works  
Tech Sales Inc.  
Tri-Power MPT  
United States Gear  
Von Ruden Mfg.  
Wedin International  
West Industries Inc.  
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 Commercial Gear &  
 Sprocket  
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 Custom Gears, Inc.  
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 Dabko Industries Inc.  
 Davall Gear Co. Ltd.  
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 Dearborn Gear & Tool Co.  
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 GA-Heartland Machine  
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 Gear Products Inc.  
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 Gerhardt Gear Co.  
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 Globe Gear Co.  
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 Great Taiwan Gear Ltd.  
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 Hamer Gear  
 Hamilton Gear  
 Hand Screw Machine  
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 Involute Tooling Corp.  
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 Jackson Gear Co.  
 Jade Precision Gear Co.  
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 KA-Wood Gear &  
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 Koellmann Gear  
 Kreiter Geartech  
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 Link Gear & Machine  
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 Overton Gear & Tool  
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 Metal Inc.  
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American Metric Corp.  
American Precision Gear  
Aplus Engineering Inc.  
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Bengal Industries  
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Cone Drive Operations  
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Davall Gear Co. Ltd.  
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Falk Corp.  
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 Reliance Gear Co. Ltd.  
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 Gear Works, Inc.  
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Mostar Gear & Machine  
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Oliver Gear, Inc.  
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Keystone Powdered Metal  
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Lawler Gear Corp.  
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M.J.H. Gear & Tool Co.  
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Micromatic Textron  
**Midwest Gear Corp.**  
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Milwaukee Gear Co.  
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Moore Gear Mfg. Co.  
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Mostar Gear & Machine  
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OEM Industries Inc.  
Oliver Gear, Inc.  
Omni Gear & Machine  
O'Neill Gear  
**Ontario Drive & Gear**  
Orlandi Gear  
Overton Gear & Tool  
Patterson Corp. & Machine  
Penn Machine Company  
Pennsylvania Gear Corp.  
Penntech  
**Perry Technology Corp.**  
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Houston  
Philadelphia Gear Corp.—  
King of Prussia  
Precipart Corp.  
Precision Gear Co.  
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Precision Gears, Inc.  
Process Industries  
Production Gear & Broach  
Progressive Engineering  
Pulley Manufacturers Inc.  
**The Purdy Corporation**  
R.L. Wagner & Assoc.  
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Rawling Gear Inc.  
Reef Gear Mfg. Inc.  
Reliance Gear Corp.  
Riley Gear Corp.  
Riverside Spline & Gear  
Rj Link International  
Ronson Gears Pty. Ltd.  
Rush Gears Inc.  
Sales Consultants  
Santasalo North America  
Satellite Gear  
Schafer Gear Works, Inc.  
SEW-Eurodrive  
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STD Precision Gear  
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**Tifo Gage & Gear**  
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Trojon Gear Inc.  
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Action Gear & Broaching  
The Adams Company  
Adobe Precision Gear  
Aero Gear  
Akron Gear & Engineering  
Allied Devices Corp.  
**Allied Gear Co.**  
American Gear & Engineering

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American Metric Corp.  
American Precision Gear  
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Bestmetal Corporation  
Blanchat Machine Co.  
Boonville Mining Services  
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Brewer Machine & Gear  
Buckeye Gear Co.  
Burgess-Norton Mfg. Co.  
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Capstan Atlantic  
Carbon City Products  
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Fairfield Mfg. Co.  
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Gear Works, Inc.  
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Gerhardt Gear Co.  
Global Gear  
Globe Gear Co.  
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Hamer Gear  
Hamilton Gear  
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HCI Supply  
Industrial Supply Co.  
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Involute Tooling Corp.  
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Jennings Machine & Gear  
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Lawler Gear Corp.  
Link Gear & Machine  
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Chandler Machine Co.  
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Trojon Gear Inc.  
United States Gear  
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CIRCLE 189

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Forest City Gear  
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Rush Gears Inc.  
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Satellite Gear  
Schafer Gear Works, Inc.  
Seitz Corporation  
SEW-Eurodrive  
Shanthi Gears Ltd.  
Springer Company Inc.  
Standard Industrial Products Co.  
STD Precision Gear  
Stock Drive Products/  
Sterling Instrument  
Suda International Gear Works  
Tracey Gear & Machine  
Tri-Power MPT  
Trojan Gear Inc.  
UFE Incorporated  
Wedin International

**Other Gears & Drives**

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Akron Gear & Engineering—*General Machining*  
American Gear & Engineering—*Clutches*  
Bevel Gears (India)—  
*Face Clutches, Custom-Made Gear Boxes*  
Bonfiglioli Riduttori—  
*Planetary Gear Boxes*  
Boonville Mining Services Inc.—*Fabrications*  
Capitol Stampings Corp.—  
*Steering Sectors*  
Commercial Gear & Sprocket—*Law*

*Clutches, Large Diameter Gears*  
Cone Drive Operations—  
*Double-Enveloping Worm Gears*  
**Crown Gear B.V.—Face Gears, Cylkro Gears**  
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Custom Gears, Inc.—  
*Miter Gears, Cut Plastic Gears*  
Davall Gear Co. Ltd.—  
*Spiradrive® (Spiroid®)*  
East Point Foundry—  
*Custom Gear Casting*  
Fairfield Mfg. Co.—  
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*Gearbox Rebuild & Recondition*  
Fleet Tools Ltd.—  
*High-Precision Gearheads*  
Flender-Graffenstaden—  
*High Speed & Epicyclic*  
Franke Gear Works Inc.—  
*Double Enveloping Worm & Gear Sets*  
Gajra Gears Ltd.—*Gear Transmissions*  
Gear Company of America—*Evoloids*  
Geartronics Industries—  
*Generated Pinion Rod*  
General Gear Corporation—  
*High Helix Gears, Speedometer Gears & Rotors*  
Globe Gear Co.—*Metric Gears*  
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J&E Hofmann Engineering—  
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Nixon Gear—*Ground Gears*  
Ohio Gear—  
*Transmissions*  
P.T. International Corp.—  
*Ground Gears*  
Philadelphia Gear Corp.—  
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—*Curvic Couplings*  
Snow-Nabstedt Power Transmission—  
*Planetary Gear Drives*  
Standard Industrial Products Co.—  
*Rotors*  
STD Precision Gear & Instrument—  
*Special Forms*

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Sussex Gear Company—  
*Linear Actuators*  
Teijin Seiki/NIMAC America—  
*Cycloidal Drives*  
Trogetec Inc.—  
*Cycloidal Gears & Harmonic Gearing*  
Waterjet Connection/  
Richel Inc.—  
*Abrasive Waterjet Machining*  
Westech Gear—  
*Specials*  
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*Film Sprockets*  
Zero-Max, Inc.—  
*Torque Limiters*

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**Basic Machine Tools**  
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Liebherr/Sigma Pool  
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Sales Consultants  
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WMW Machinery

**Broaching Machines**

American Broach & Machine Co.  
American Machinery  
**Colonial Tool Group Inc.**  
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**Fässler Corp.**  
General Broach & Eng.  
Jack Dustman & Assoc.  
Kingsford Broach & Tool  
Miller Industrial Service  
**National Broach & Machine Co.**  
The Ohio Broach & Machine Co.  
Oswald Forst GmbH  
Sales Consultants  
Ty Miles Inc.  
U.S. Broach & Machine

**Burnishing Machines**

ITW Heartland  
M&M Precision Systems  
Spline Gauges Ltd.

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American Machinery



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Gleason-Pfauter GmbH  
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**Mitsui Machine  
Technology**  
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Gleason-Pfauter GmbH  
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**The Gleason Works**  
GMI  
**Holroyd Machine**  
Jack Dustman & Assoc.  
Progressive Technologies  
Redin Corporation  
**SU America Inc.**  
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Machinery  
Charnilles Technologies  
Easco-Sparcotron  
Hansvedt Industries Inc.  
KKG International Corp.  
Makino  
Mecatool USA Ltd.  
Okamoto Corp EDM Div.  
Raycon Corporation

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Machines**

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Machinery  
**Bourn & Koch Machine  
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**Gleason-Pfauter-Hurth  
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**Liebherr/Sigma Pool**  
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Miller Industrial Service  
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Furnace  
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Pacific Industrial Furnace  
Pillar Industries  
Quench Press Specialists  
**Radyne Corporation**  
**Surface Combustion Inc.**  
TOCCO Inc.  
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V.T.M. Co. Ltd.

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GA-Heartland Machine  
Tool  
Gleason-Pfauter GmbH  
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The Gleason Works**  
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Lees Bradner Div.  
Fayscott Co.  
**Liebherr/Sigma Pool**  
**Mitsubishi Machine Tool**  
**Mitsui Machine  
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**National Broach &  
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Bluegrass Precision  
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FGT Gage & Systems Inc.  
Gleason-Pfauter GmbH  
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Kokusai Inc.  
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**Roto-Technology, Inc.**  
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Mitts & Merrill L.P.  
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**Gleason-Pfauter-Hurth  
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Lapmaster International  
**Liebherr/Sigma Pool**  
Oerlikon Geartec AG

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**Basic Machine Tools**  
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Machinery  
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**Holroyd Machine**  
**ITW Heartland**  
Klingelberg Söhne GmbH  
Krautkramer Branson  
**Liebherr/Sigma Pool**  
**M&M Precision Systems**  
**Mahr Corporation**  
Miller Industrial Service  
Oerlikon Geartec AG  
Ono Sokki Technology  
Precision Gage Co.  
**Profile Engineering, Inc.**  
Spline Gauges Ltd.  
United Tool Supply

**Milling Machines**

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**Bourn & Koch Machine  
Tool Co.**  
Gleason-Pfauter-Hurth  
**Mitsubishi Machine Tool**

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Presses**

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**Liebherr/Sigma Pool**  
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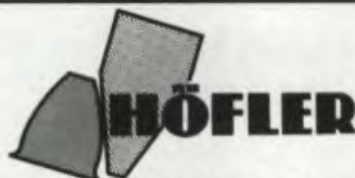


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CIRCLE 193



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CIRCLE 192

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USACH Technologies—  
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Zhuhai Intercontinental Pulleys Ltd.

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Cornell Forge Co.  
Dabko Industries Inc.  
Dura-Bar  
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Elmass North America  
Forging Specialties  
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Globe Gear Co.  
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Howard's Machine Shop  
J&E Hofmann Engineering  
Lovejoy Steel—Cincinnati  
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Process Industries  
Pulley Manufacturers Inc.  
Qualicast Corp.  
Rush Gears Inc.  
Sales Consultants  
Stock Drive Products/  
Sterling Instrument  
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Wes-Tex Gear Inc.  
Wohler Corporation  
Worrall Grinding Co.

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Bengal Industries  
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Pulley Manufacturers Inc.  
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Metal Powder Industries Federation  
Rush Gears Inc.

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Forging Specialties  
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J&E Hofmann Engineering  
Latrobe Steel Company  
Lovejoy Steel Company—  
Charlotte  
Lovejoy Steel—Cincinnati  
Lovejoy Steel Company—  
Streetsboro  
Lovejoy Steel Company—  
York  
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Moore Machine & Gear  
Pulley Manufacturers Inc.  
Rush Gears Inc.  
Scot Forge  
Trogetec Inc.  
Wes-Tex Gear Inc.

**Other Materials**  
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Forging Specialties—Ring Forgings  
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CIRCLE 270



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 Software Engineering  
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 Sussex Gear Company  
**Tifco Gage & Gear**  
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 Van Gerpen-Reece  
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 Caterpillar Industrial  
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 Chardam Gear Co.  
 The Cincinnati Gear Co.  
 Columbia Gear Corp.

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 Dayton Gear & Tool  
 Fairfield Mfg. Co.  
 Federal Gear Corp.  
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 Ashot USA Inc.  
 Blanchat Machine Co.  
 Caterpillar Industrial  
 Products Inc.  
 The Cincinnati Gear Co.  
 Commercial Steel Treating  
 Contour Hardening, Inc.  
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 Dynamic Metal Treating  
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 Metlab Co.  
 Mill Max Tools Pvt. Ltd.  
 Milwaukee Gear Co.  
 Modern Industries Inc.  
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 Moore Gear Mfg. Co.  
 Mr. Gears, Inc.  
 Multi-Arc Inc.  
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 Nitrex Metal Technologies  
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 Paulo Products Company  
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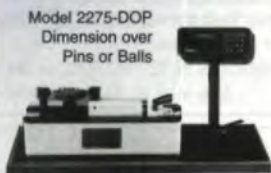
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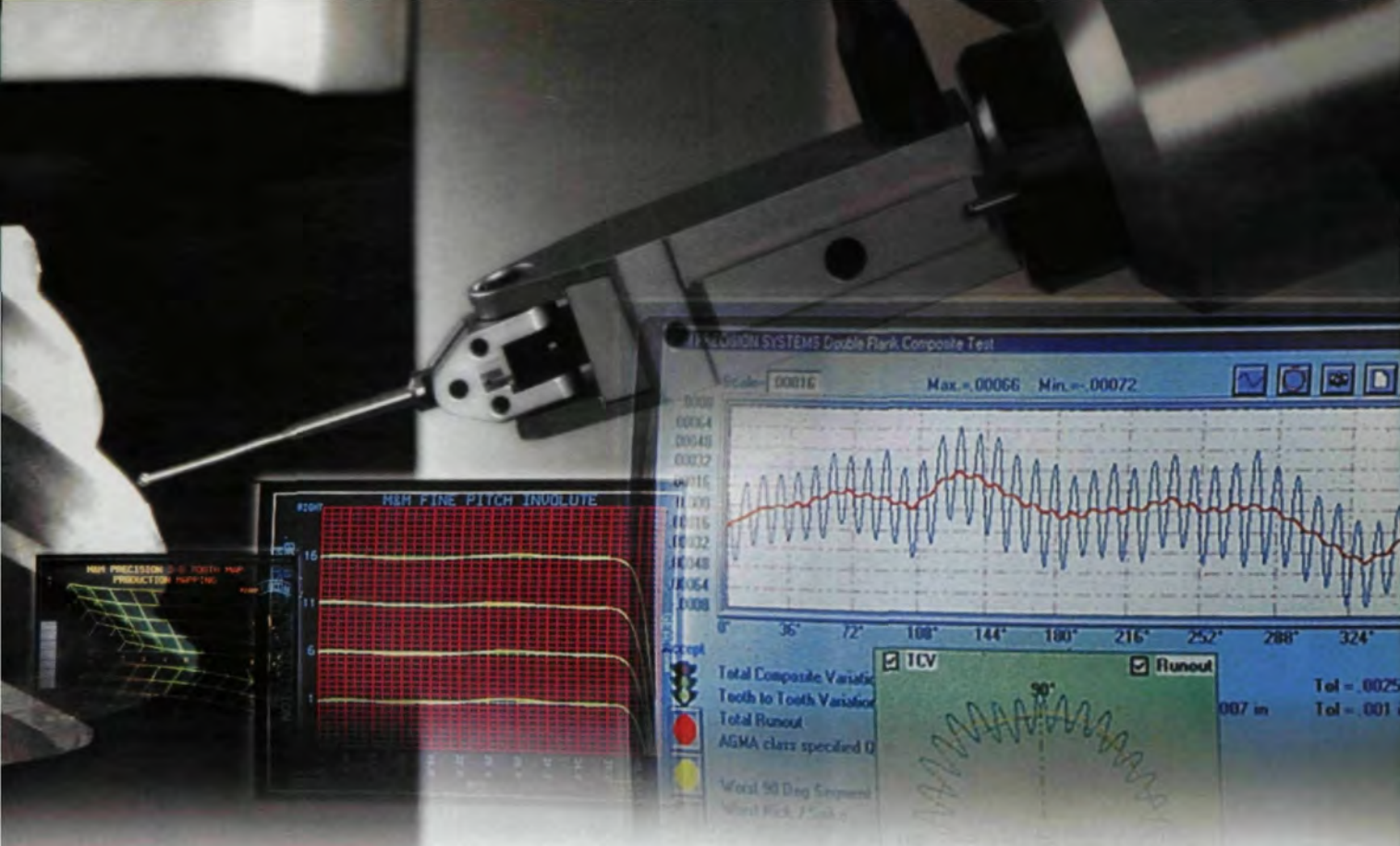
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Australia Ph: (61) 49593573  
Fax: (61) 49504538

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*www.appliedprocess.com*

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Baldor Motors & Drives  
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 BP 80067  
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 France  
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 michel.octruie@cetim.fr

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Ciateq, A.C.  
 Mechanical Transmissions Div.  
 Retablo 150  
 Queretaro 76150  
 Mexico  
 Ph: (52) 4216808  
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 rodlop@ciateq.mx

The Cincinnati Gear Company  
 5657 Wooster Pike  
 Cincinnati OH 45227  
 Ph: 513-271-7700  
 Fax: 513-271-0049  
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Circle Gear & Machine Company  
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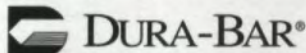
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691 N. Squirrel Rd.  
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United Kingdom  
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Fax: (44) 1484-401618

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6556 Arrow Drive  
Sterling Heights MI 48314  
Ph: 810-254-4600  
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122 Pembroke St.  
P.O. Box A  
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sales@csaw.com  
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5505 Concorde  
Detroit MI 48211  
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bmfroats@mnsi.net

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dianamic@aol.com  
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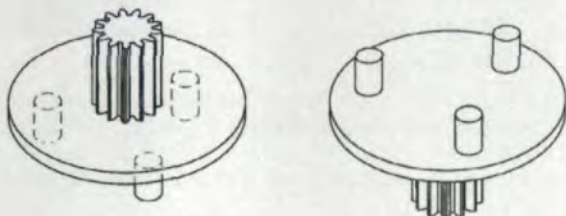


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CIRCLE 121

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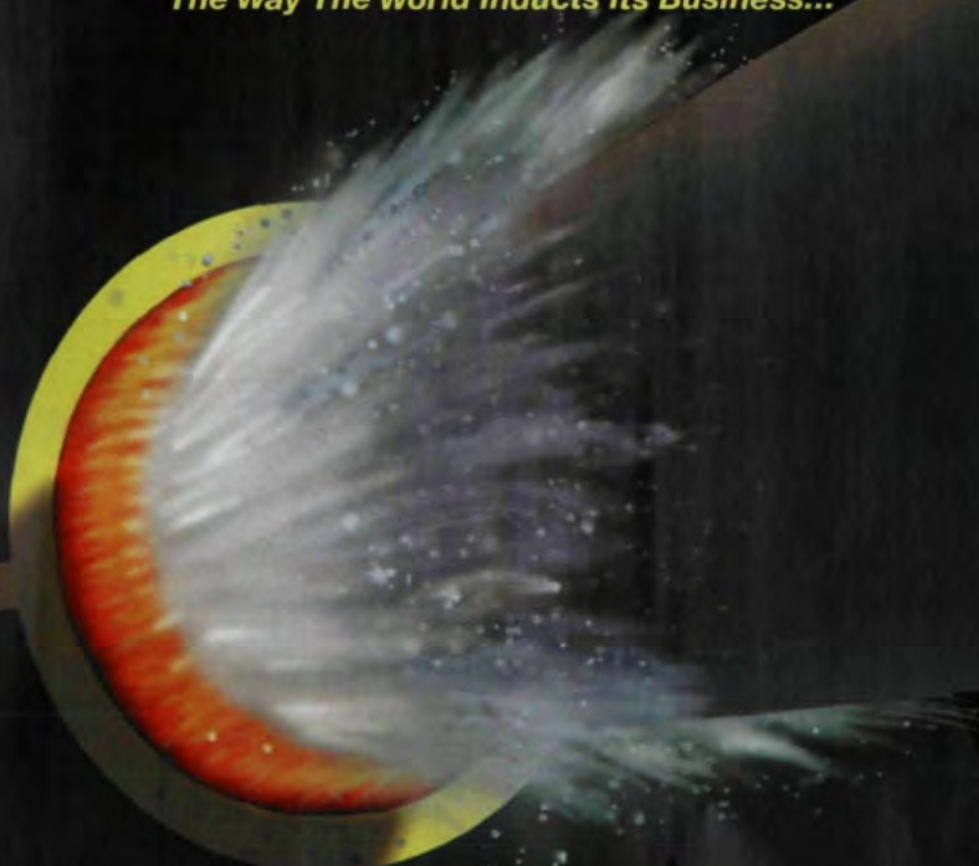
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# The Basics of Gear Metrology and Terminology Part II

Edward Lawson  
Senior Applications Engineer, Mahr Corporation

*In the last section, we discussed gear inspection; the types of errors found by single and double flank composite and analytical tests; involute geometry; the involute cam and the causes and symptoms of profile errors. In this section, we go into tooth alignment and line of contact issues including lead, helix angles, pitch, pitchline runout, testing and errors in pitch and alignment.*

## Tooth Alignment

**Helical Lead and Helix Angles.** The lengthwise profile of an involute helicoid gear tooth is a simple helical lead. Testing of this parameter is still commonly referred to as lead testing despite the changes in the AGMA standard, which now more precisely terms this parameter as tooth alignment. The helical lead of a gear tooth has the same geometry as the helical lead of a screw thread. It is defined by the axial advance (the lead) per 360 degrees of rotational displacement upon a cylinder. For spur gears, the lead is infinite (Fig 1).

While the typical screw thread embodies many complete helical leads, the typical gear tooth embodies only a fraction of one lead. It is, therefore, more difficult to visualize the helical geometry of the gear tooth. However, the helical gear tooth, which appears to be simply tilted at an angle to the gear axis is, in fact, both tilted and wrapped around a cylinder, thereby producing the helical geometry. If a gear tooth alignment is considered by unwrapping the pitch cylinder of the gear onto a flat surface, it will then truly appear as a straight line inclined at an angle to the gear axis. The helix/lead angles will be determined

by the relationship of the diameter of the cylinder and the length of the lead specified for the tooth.

The helix angle is found between the tooth alignment and the gear axis while the lead angle is found between the tooth alignment and the transverse plane. It is common for these terms to be incorrectly used interchangeably. A close visual examination of a helical gear tooth will reveal that as the cylinder diameter increases, the helix angle also increases. It is important to note that this phenomenon is a function of differing diameters while the lead remains the same for all diameters of the gear tooth.

**Testing Tooth Alignment.** This characteristic of helical gear teeth allows for a simple tooth alignment test not unlike the generative tests described in Part I (*Gear Technology*, Sept./Oct.1998). In this case, a sensitive probe is brought into contact with the tooth at any diameter and is then carried in a direction parallel to the gear axis while the gear is rotated. The gear rotation must be in a direct relationship with the axial motion of the probe according to the lead of the gear. If the gear was to be rotated a complete 360 degrees, the probe would move axially a distance equal to the lead. If the tooth alignment being tested is perfect, the sensitive probe will measure no error.

Most such generative tooth alignment testing instruments use some sort of helix guide or sine bar arrangement. CNC devices test tooth alignment by controlling the movements of a rotary axis (spindle) and vertical slide that are not connected by any mechanical components. They are each caused to move under computer control in the same fashion as the classic mechanical instrument

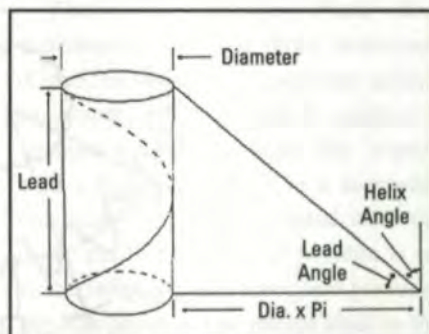


Fig. 1 — Tooth alignment (formerly "Lead"). Courtesy of AGMA.

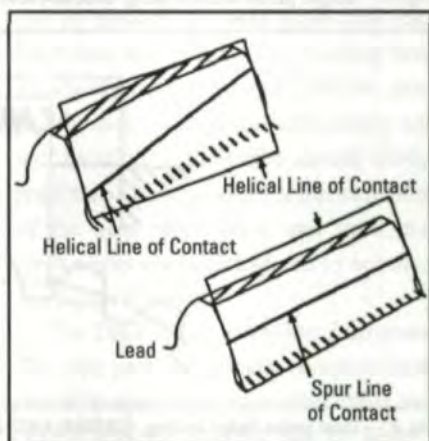


Fig. 2 — Line of contact. Courtesy of AGMA.

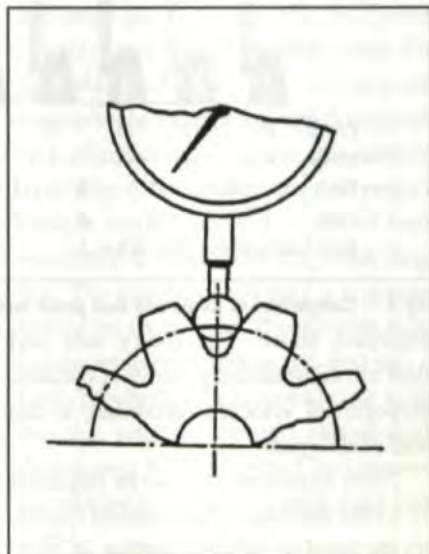


Fig. 3 — Pitchline Runout. Courtesy of AGMA.

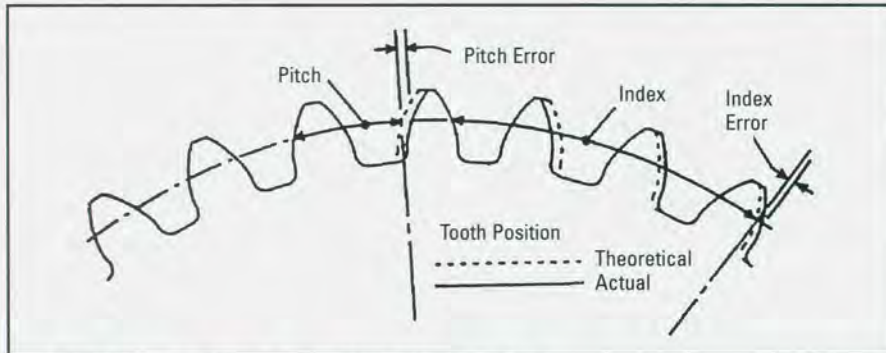


Fig. 4 — Pitch error and index error. ©AGMA/ANSI 2000-A88.

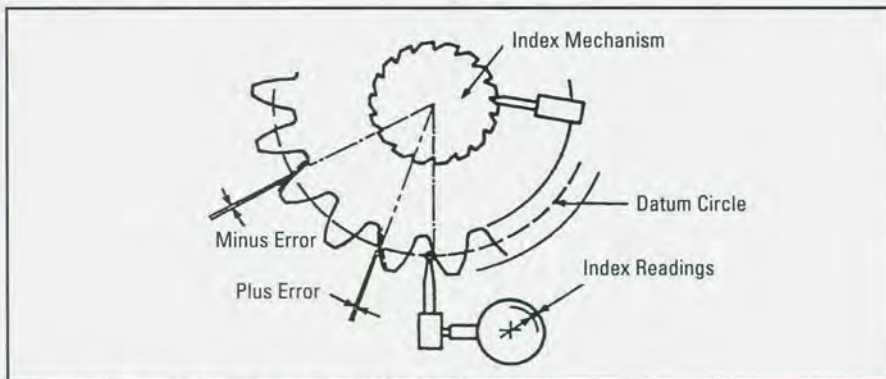


Fig. 5 — Single probe index testing. ©AGMA/ANSI 2000-A88.

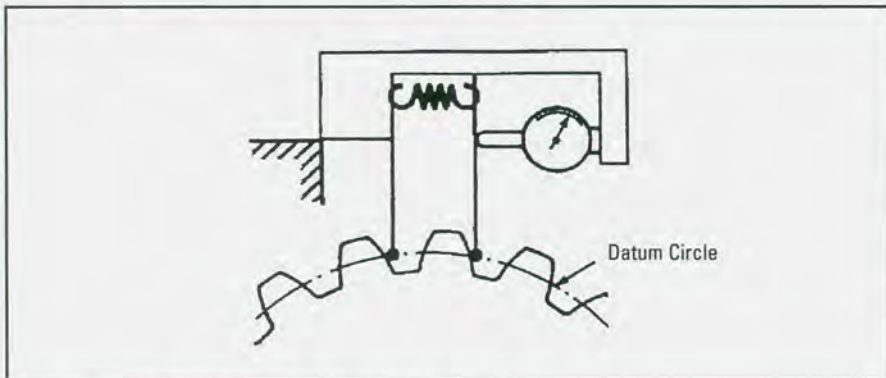


Fig. 6 — Dual probe index testing. ©AGMA/ANSI 2000-A88.

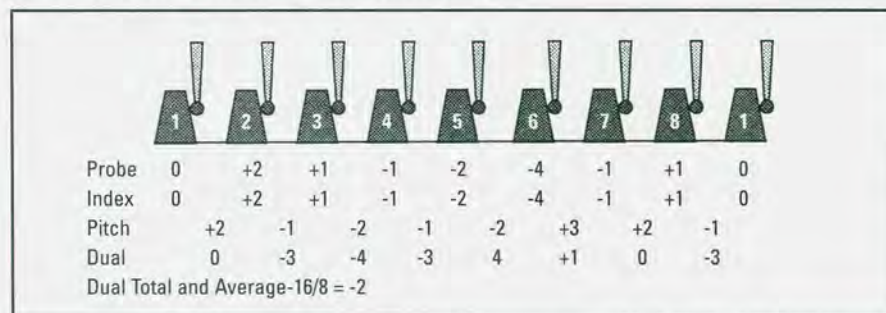


Fig. 7 — Comparison of single and dual probe test measurements. ©AGMA/ANSI 2000-A88.

described above. The rotary axis and slide are commanded to move at constant proportional velocities according to the lead of the gear.

Tooth alignment can also be inspected by CMM machines. This method considers the involute helicoid surface in rectangular coordinates, a considerably more

complex process than the classic generative method described above and an uncommon one.

*Types of Tooth Alignment Errors.* The primary concern associated with tooth alignment errors is their adverse effect upon gear strength and durability. Tooth strength ratings are normally calculated

assuming that the torque load will be applied uniformly across the full face width of the gear. Tooth alignment errors can reduce the region of contact between mates, significantly increasing both the structural stresses and surface contact stresses on the tooth, reducing the gear's actual strength and durability values.

A common category of tooth alignment error is slope error. This type of error causes increased loading on the tooth ends, the place they are least capable of carrying increased stress.

Tooth alignment errors are not commonly associated with noise problems. A gear can have a substantial tooth alignment error and still produce excellent conjugate action if the involute geometry in the reduced region of mesh is of good quality. Tooth alignment errors can, however, increase noise levels for a given transmission error signature by increasing the loading on the gear teeth during meshing action.

*Causes of Tooth Alignment Errors.* Machine tool alignment problems produce tooth alignment errors that are consistent when one observes teeth located at various positions around the gear. Such tool alignment errors made in a direction perpendicular to tool feed produce tooth alignment errors of equivalent direction and slope on opposite tooth flanks. Errors parallel to the tool feed direction produce tooth alignment errors of different direction and slope on opposite tooth flanks.

Errors in gear blank accuracy or mounting produce tooth alignment errors that vary when one observes teeth located at different positions around the gear. One category of accuracy or mounting error is simple eccentricity wherein the axis of the gear feature is parallel with, but offset from, the axis of rotation. The other category of accuracy or mounting error is axial runout (a.k.a. wobble or cranking) wherein the axis of the gear feature is not parallel with the axis of rotation.

Both categories of accuracy or mounting error cause tooth alignment errors varying in slope in a sinusoidal pattern around the gear. However, only axial runout will adversely affect the performance of the gear. Simple eccentricity produces only apparent tooth alignment

errors (as was the case with involute profiles). It is usually visible only in cases of high helix angles and large face widths. Simple eccentricity has no measurable effects upon the accuracy of spur gear tooth alignment. Segregation of the two categories of accuracy or mounting error is moderately complex and is not commonly done.

#### Line of Contact

As mentioned in Part I, gear teeth only contact one another within the plane tangent to their base cylinders. An interesting feature of the involute helicoid is that its intersection with the plane of action is always a straight line, so involute helicoid gear teeth are always contacting one another along straight line elements within their surfaces called lines of contact.

A line of contact for a spur gear is the same as a tooth alignment line. For helical gears, the line of contact is inclined across the tooth face at an angle (Fig. 2). It is thereby affected by both errors of involute profile and tooth alignment, which may combine in either a cumulative or compensatory fashion.

Observation of line of contact traces may permit acceptance of a gear with significant errors which are compensatory in nature. It is important in such cases to recall that the process is not operating properly even though the product is OK to ship. As a composite of the involute profile and tooth alignment parameters, the line of contact usually does not correlate well with process variables.

The line of contact is always inclined to the helix angle at the base circle of the gear. It is possible to test the line of action by moving a sensitive probe within the plane of action at the base helix angle while in contact with the tooth flank. Mechanical testers have been developed to do this test. CNC testers can also be programmed to make this test move. Line of contact testing is not common and AGMA standards do not tolerance it.

#### Pitchline Runout

Pitchline runout testing is an old and simple method of limited value. The test gear is mounted on a spindle free to rotate. A sensitive probe is brought into orientation to measure in a radial direc-

tion. A ball or other shape is selected to make contact with both flanks of the gear near the pitchline or other reference diameter. The probe is then moved into the tooth space until it reads zero while contacting both flanks and that position is set to be repeated on subsequent tooth spaces. The probe is then moved clear so the gear can be rotated, then moved back into the next tooth space. This process continues until measurements are taken on all the tooth spaces of the gear. The assumption is that a gear with no radial runout will produce all zero readings on the probe (Fig. 3).

Since this test is a double flank radial direction test, it relates only indirectly to gear performance. Pitchline runout test data can reveal radial eccentricity or out-of-round errors which produce gear transmission error. Certain manufacturing processes (e.g., shaving) often produce gears with significant angular errors that cannot be detected by double flank testing.

Pitchline runout errors typically result from gear blank geometry problems or gear blank mounting errors. These mistakes cause cyclic angular tooth position errors comparable to tooth index errors with respect to both duration and amplitude, variation of backlash (which becomes a major consideration in applications where minimum backlash is required), and apparent profile slope variation (which has little effect on gear performance).

#### Pitch and Index

**Index Error.** This is the displacement of any tooth on a gear relative to any other tooth, measured in a tangential direction. Index errors are commonly called cumulative pitch errors. This parameter is ideal for observing tooth position errors of longer duration such as eccentricity (one cycle per revolution) or out-of-round (Fig. 4).

**Pitch Error.** This is the displacement of any tooth on a gear relative to an adjacent tooth, measured in a tangential direction. Commonly called adjacent pitch error, this parameter is ideal for observing tooth position errors of shorter duration associated with localized meshing conditions. Less commonly used is the spacing error parameter—the difference

between the pitch value for one pair of teeth and the pitch value for the adjacent pair of teeth.

**Pitch Testing.** Testing tooth location parameters is done with two types of instruments: the single probe and the dual probe. The single probe is usually seen as providing more reliable information, especially in the presence of larger numbers of teeth. However, the dual probe may be superior for very large diameter gears.

**The Single Probe Pitch Test Instrument.** The single probe instrument (Fig. 5) compares tooth positions against an accurate dividing head. In practice, a sensitive probe is brought into contact with a tooth, measuring in a tangential direction. The gear is rotated until the probe reads zero, and that position is set as the reference for subsequent positioning. The probe is then moved clear so the gear can rotate to the next tooth position. The probe is then brought back into contact to measure the next tooth until each tooth has been tested. The dividing head accurately rotates the gear  $360^\circ/N$ , placing the next tooth at its theoretically correct location so the probe should always read zero. This provides a direct reading of the gear index error, and from that, pitch errors can be calculated by subtracting adjacent values.

**The Dual Probe Pitch Test Instrument.** The dual probe instrument compares tooth positions against the pitch of an initial, randomly selected adjacent pair of teeth. It measures the differences between that pair and other pairs of teeth. The two probes consist of a fixed and movable probe. The moveable probe measures displacement relative to the fixed probe, which establishes a reference point against the adjacent tooth (Fig. 6). In practice, the dual probe is brought into contact with a pair of teeth, measuring in a generally tangential direction. The gear is rotated until it is stopped against the fixed probe. The movable probe is then set to read zero on this first pair of teeth. The dual probe is moved clear so the gear can rotate to the next pair of teeth, then moved back to contact and measure the relative pitch until all teeth have been tested.

To calculate pitch errors from dual probe data, it is first necessary to find the

average value for all the relative pitch measurements. All measurements would be equal to that average value if the gear were perfect. Subtracting the average from measured values gives the pitch errors for the gear. Successive summation of those pitch error values produces the index error values for the gear. It is a common misconception that the dual probe test instrument will directly measure pitch errors. It is important to understand that the average

value must be subtracted before pitch error values are produced.

Figure 7 displays the various tooth location parameters. It shows an eight-tooth gear which has been unwrapped into a straight line. A sensitive probe is brought into contact with the first tooth and zeroed. It is then moved to the theoretically correct position on each subsequent tooth and the tooth position is measured as shown in the first line of data.

This data shows that tooth #2 is 2 increments out of position to the right (+2), and that tooth #5 is 2 increments out of position to the left (-2). Notice that the index errors relative to tooth #1 are measured directly by this procedure, which is the equivalent to a single probe test. The worst index error (total error) is 6 increments and occurs between tooth #2 (measured +2) and tooth #6 (measured -4). The index error is not given a + or - sign since that would depend upon the direction taken around the gear from #2 to #6.

Pitch error data can be derived from the index data by successive subtraction. For example, the +2 pitch error of tooth pair 7 and 8 is found by subtracting the #7 value (-1) from the #8 value (+1). The error is + since the teeth are too far apart rather than too close together. A possible set of dual probe values for this gear is also provided. The average is found to be -2. Subtraction of that average value from the actual measurements will produce the same pitch values as the single probe approach. Successive summation of pitch values will produce index values. The +1 index value for #8 equals the #7 index value (-1) plus the pair 7 and 8 pitch value (+2).

#### *The Causes of Pitch and Index Errors.*

Pitch and index errors are both caused by three things: problems with machine tools, cutting tool problems and gear blank and mounting errors.

Kinematic errors in machine tools, including simple looseness, can generate pitch errors when used with certain design processes that transfer the imperfections of the cutting tool directly to the work piece such as hobbing a gear with a multiple thread hob where the number of hob threads is a factor of the number of teeth on the gear. That process will result in the thread spacing errors of the hob transferring directly to pitch errors on the gear. Index errors may be associated with processes that do not control the angular positioning of the gear during cutting (e.g., shaving), but another possible cause is runout in a change gear, causing the production gear to accelerate/decelerate its rotation during cutting.

Gear blank and mounting errors include both eccentricity and out-of-

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round problems and are the primary cause of index errors. A gear that is otherwise perfect except for being eccentric by .005 mm will display a sinusoidal pitchline runout of .010 mm. This will result in an acceleration/deceleration of the points of contact with the mating gear teeth along the line of action of .010 mm. Eccentricity in the gear blank can produce pitch errors in an apparent sinusoidal pattern that is especially noticeable when the number of teeth is relatively small. However, this apparent pitch error will not adversely affect gear meshing action performance. As with the involute profile and helical tooth alignment, if a proper observation of simple eccentricity is applied to the pitch data in an appropriate fashion, the resulting modified pitch error data is valid for observations of localized tooth meshing conjugacy.

*Problems Arising From Pitch and Index Errors.* Pitch errors lead to gear noise by being a significant source of transmission error. However, the noise produced by a pitch error is typically less objectionable than the equivalent noise produced by profile errors. This is because some tooth pairs show plus pitch errors and others minus errors, causing a less consistent transmission error characteristic than profile type errors. Index errors are not often associated with gear noise trouble, though the cyclic modulation they can produce may be noticeable. It is also possible for the increased loading of a planetary set caused by index errors to cause increased noise levels. Index errors are also a common cause of ghost harmonics, an error which may appear in index test data if they occur at lower than mesh frequencies.

Both pitch and index errors lead to strength problems since dynamic loading promotes fatigue. The shock load that occurs when mispositioned teeth enter mesh adversely affects the strength of those teeth because the dynamic stresses involved will be higher than the tooth was designed to take. Localized pitting and other durability problems can also occur in the region of initial contact between mispositioned mating teeth. Other strength and durability problems associated with index errors include

increases in dynamic loading in high-power, high-speed gear drives due to cyclic changes in rotational velocity and backlash variation if the problem is caused by pitchline runout. Index errors also significantly reduce load sharing in planetary gear sets, increasing the importance of single flank testing, either composite or analytical, on gears produced for automotive style automatic transmissions as those are often finished by shaving.

Finally, index errors can degrade the performance of gears that must maintain timing relationships. This includes positioning applications such as robots and media movement applications such as printing presses. ⚙

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CIRCLE 254

# Changes In The Gear Industry

## PFAUTER-MAAG CUTTING TOOLS RECEIVES ISO 9001 REGISTRATION

Pfauter-Maag Cutting Tools is now registered as an ISO 9001 company under KPMG Quality Registrar. This registration makes Pfauter-Maag Cutting Tools one of the first gear cutting tool manufacturers to meet this important international quality standard. This ISO 9001 certification stems from the company's commitment to meet or exceed the world's highest quality and consistency standards.

"Our facilities here and abroad are among the best in the world for the design and manufacture of advanced gear cutting tools," said Pfauter-Maag Cutting Tools quality manager Jim Argyle. "Management commitment, continuous improvement, investment in training, state-of-the-art equipment and other critical resources are necessary to meet or exceed the toughest quality and performance standards of our customers. That's what differentiates us from other suppliers."

## NORD GEAR CORPORATION EXPANDS SALES AND ASSEMBLY OPERATIONS IN NORTH AMERICA



Nord Gear announces the opening of a 40,000 sq. ft. sales and assembly facility in Brampton, Canada; a 20,000 sq. ft. sales and assembly facility in Charlotte, NC and regional sales offices in Auburn Hills, MI. The addition of these facilities will support Nord Gear's efforts to significantly reduce lead times and improve overall customer service. Nord Gear also announces plans to break ground for an additional 30,000 sq. ft. of manufacturing space and 20,000 sq. ft. of administrative office space at its North American headquarters in Waunakee, WI.

## NATIONAL BROACH WELCOMES NEW PRESIDENT AND CHIEF OPERATING OFFICER

Carmen Calabrese, Ph.D. has joined National Broach and Machine Co. as President and Chief Operating Officer. His diverse experience in upper-level management, marketing, sales, business development, manufacturing and business education will be an asset to National Broach and Machine Co. Dr. Calabrese's extensive experience with multi-national companies including Murata Machinery; Asea Brown Boveri (ABB) Ltd.; Robert Bosch, GmbH; the Midland-Ross Corporation; and E.I. DuPont De Nemours, Inc., provides him with a keen understanding of today's global marketplace. He holds a Ph.D. from the University of Pennsylvania in Materials Science and Engineering; an MBA in Marketing/Management from Eastern Michigan University and a BS in Metallurgical Engineering from Drexel University.

Dr. Calabrese has stated four goals for National Broach, each interrelated. First, National Broach must create an environment where employees are both committed and dedicated. Secondly, the company must provide quality products that meet and exceed customer expectations. The third goal is to generate and maintain customer satisfaction and loyalty. Finally, the natural outcome of the previous goals is a return on investment to the stockholders.

## EMCO MAIER CORPORATION NAMES NEW PRESIDENT

Robert J. Pernsteiner has been named President of EMCO Maier Corp. He comes to the post with over 30 years of experience in the machine tool industry. "I believe in being a hands-on team player to achieve EMCO Maier's long-term objectives," said Pernsteiner. "I am charged with using my experience to effectively manage our EMCO Maier team. Our primary goal is to steadily increase U.S. market share for EMCO Maier."

Pernsteiner began his professional career with Kerney & Trecker Corp., spending 18 years managing such diverse areas as production planning, purchasing and materials. In 1986 he moved over to Okuma where, over 12 years, he advanced in the areas of operations and customer service management, materials management, sales and marketing. "Bringing EMCO Maier to a leadership position in the American marketplace presents an exciting and fresh challenge for me," said Pernsteiner. "It is one I look forward to with great anticipation. I have no doubt that our staff has the commitment and fortitude necessary for the task at hand."

## HURCO MACHINE TOOLS NAMES NEW PRESIDENT



Richard Blake

Hurco Machine Tool Products has announced the appointment of Richard Blake as President, with responsibilities for sales and service in North America and Europe. Trained as a tool and die maker, Blake brings more than 20 years of experience in the machine tool industry. With Hurco for the last 10 years, Blake most recently served as Managing Director of Hurco Europe in High Wycombe, England. His foremost goal now is to grow Hurco in North America.

"We are now the number two supplier of vertical machining centers to Germany. Throughout Europe over the last four years, we have enjoyed significant growth and increased market share. We need to carry that success into North America. To do this, we must focus on a very high level of support for our key distribution and to continue to exceed our customers' needs in all aspects in their machine tool requirements." ❁

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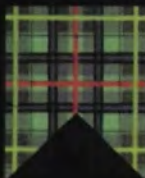
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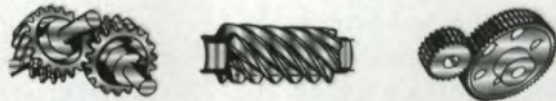
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CIRCLE 154



### New Gear Deburring Machines From American Wera

American Wera, Inc. has added three new machines to its line of gear-related machining solutions. The ZEM-180 is a new spur gear deburring machine designed to be easily adapted to new or existing gear hobbing machines. The KEM 200 and KEM 500 are bevel gear deburring machines with five-axis CNC capabilities.

Designed for limited floor space (55"x51"), the ZEM-180 is equipped with an integrated dual-gripper pick-and-place station. Positioning this machine next to a gear hobber, you would turn the hobbing machine work station into an automatic production cell. The five-axis bevel gear deburring KEM machines are designed to deburr both toe and heel gear teeth in a single chucking. The CNC-controlled tool axis of the KEM machines swivels automatically to its programmed position. The KEM-200 can also be equipped with a ring loader and a double-gripper pick-and-place station. The KEM-500 is a larger version of the same machine. For more information contact American Wera, 4630 Freedom Drive, Ann Arbor, MI 48108 or call (313) 973-7800.

Circle 300

### Basic Machine Tools Unveils New Gear Hobber

Basic Machine Tools, national distributor of 75 models of gear machines, announces their new WOLF model GH32-19LS gear hobbing machine. According to W.F. Wolf, CEO of Basic, "The new GH32-19LS is a unique gear hobber, the answer to many manual horizontal hobbers so popular in the past but no longer available as new machines."

The GH32-19LS is especially suited for long shaft work. Up to 5" diameter shafts can be inserted into the work table and up through a bushing in the tailstock an unlimited distance. The new GH32-19LS can cut gears up to 32" diameter and pitches to 2.5 DP. Differential gears provide helix angles to 45°. The vertical work piece axis permits better chip fall away than the old horizontal machines. For more informa-

tion contact Basic Machine Tools, P.O. Box 36276, Los Angeles, CA 90036 or call (323) 933-7191.

Circle 301

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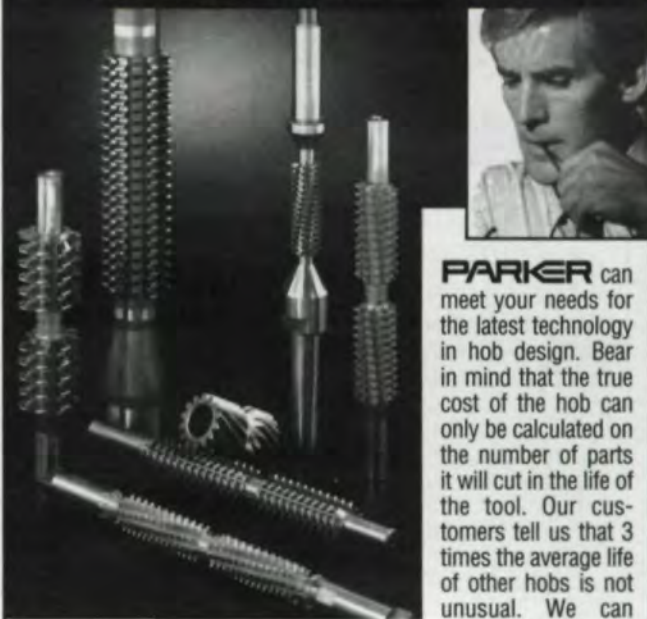
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**WORM GEARS**

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# Poetry

As we at Addendum have long known, within every gear man (and woman) lies the soul of a poet.



# In Mesh

To prove it, we present the following piece by David B. Dooner.

**Gear Technology's bimonthly aberration — gear trivia, humor, weirdness and oddments for the edification and amusement of our readers. Contributions are welcome.**

History has pegged wheels in mesh circa 2600 BC as ancient Chinese traversed the Gobi desert to see. In motion was the South Pointing Chariot to lead with epicycloidal movement at face and in feed as the ubiquitous use of gear pairs were cast to be.

Traces of gears were put in print with work of Aristotle, not as science but as machine drives for a future throttle. For fifteen centuries advancements had little to show where materials matters were mute and speeds were slow as increased revolutions spurred designers to model.

In 1694 Philip de la Hire sired the tooth type involute; years 150 later Euler fathered details with circular evolute. Abandoned is the widespread cycloidal profile; notwithstanding timepieces, tooth types anew are rendered futile as details are often dimensioned as mathematically convolute.

Nomenclature needed to number the tooth parts with pitch: diametral, circular, axial, and transverse; normally known is which. EAP, HPSTC, and tip relief are all aspects of addendum; TIF, SAP, LPSTC, fillet, root, and base define dedendum as clearance, backlash and contact ratio are added to enrich.

Different gear types make many classes too: rack & pinion, non-circular, and worm identify a few. Matters not whether cylindrical, conical, or hyperboloidal for all degenerate cases are ordered cylindroidal as spiral, spur, and straight are species sighted at a gear zoo.

To EGT or PGT is differential only in name for the motions made vary the same. So be it central, intermediate, annulus, carrier, and case; otherwise call it sun, planet, ring, arm, and base as other families of motions are made via a simple gear train.

Differential geometry is a topological tool with a goal; the objective is to minimize the amount of slide-to-roll. Euler-Savory defines surfaces of conjugate curvature and laws of gearing guarantee fundamentals of nature as freedom from efficiency is the price of the toll.

Tribology of contact can be studied to a point where friction at contact gets graphite, grease or oil to anoint. Be it centipoise, stokes, micro-reyns, saybolts, or else temperature rise at mesh must maintain before gear set melts; as EHD lubrication is an item of aim at each joint.

Experience or none consultants are available for a fee; to contract out the specs an expert will want to see. List of concerns needed prior to angle of approach so success will be a measure of market share encroach as all claim to cut cost and none are for free!

The design of a gear set requires ratings and an initial guess; to evaluate inertial, bending, contact, and thermal is shear stress. Splined, keyed, webbed, or rimmed: a choice one must make prior to determining if fatigue is critical and at stake as the decision to FEA the calculations can make a mess.

First, to fabricate a gear set requires a machine at last; CNC or not, options are to press, mill, forge or cast. EDM and RP are not mass production like generating hardware where the number of teeth in mesh can occur anywhere, so hunt a ratio and shave, hone, or burnish the finish fast.

TiN, TiAlN, and XYZ to be can wear a better rate as tooth films or tool coatings when cutting its mate. To gage the amount necessary is to measure when the film thickness is distributed evenly thin as choices PVD, CVD, or NEW can create a heated debate.

Caution, post processing produces profile modification with fluctuations in I/O relation as well as noise and vibration. Be it at a tooth mesh frequency or sideband, a runout of envelope calculations are easy and off-hand, so consider carefully as resonance can cease utilization.

To optimize is to search for numbers in vain; gear parameters are balanced with a gradient to constrain. Material may be plastic, bronze, aluminum, iron "ore" steel; as all must be considered to generate the best deal, for a favorite is found with increasing power to weight gain.

ISO—a metric that is independent of ips and cgs, along with a gear doctor that can cure mis-alignment for fps or mks. Be it hard to know when to Bhn, Rc, HSc, or mho; for it's time to trade schools when masters do not know enough to tell when to DIN, AGMA, or BS.

**David B. Dooner** is an Associate Professor in the Department of Mechanical Engineering at the University of Puerto Rico—Mayagüez. He received his doctorate from the University of Florida in 1991 and continued his studies there as a post-doctoral fellow until 1994. He is the co-author of the text *The Kinematic of Geometry: A Concurrent Engineering Approach* with Ali Seireg. Also, he is co-author of multiple patents pertaining to power transmission and gear manufacturing. He worked at the General Motors Gear Center in 1989 and in 1992 was a visiting scientist at the Mechanical Sciences Research Institute of the Russian Academy of Sciences in Moscow.

Gear photo courtesy of GEARTECH Gear Research, Analysis and Design.

**The Addendometer:** If you've read this far on the page and enjoyed it, please circle 225.

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