



UTILIZATION OF POWDER METAL AND SHOT PEENING RESIDUAL STRESS TO MAXIMIZE COST AND PERFORMANCE BENEFIT OF HIGHLY LOADED GEARING

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Management Summary

The primary benefit of powder metal (P/M) gearing is reduced cost of manufacturing, making such gearing an obvious choice for lower load applications. The perception that P/M gears are limited to only low load classes has been changing with recent developments in higher density P/M technology and advanced manufacturing techniques, such as powder forging, isostatic pressing, conventional compaction, surface densification, and activated sintering.

This paper will focus on bending fatigue strength improvements of P/M gearing from recent improvements in P/M technology combined with the established technology of shot peening. Factoring in the significant cost savings of P/M manufacturing, P/M gears with shot peening have the potential to replace higher load applications currently served by wrought gearing. This paper addresses gear applications limited by bending fatigue and not other failure modes.

Introduction

Increasing the load carrying capacity of gearing has been of principal importance since gears have been in existence. In modern day manufacturing, the cost of gear production continues to play a significant role in selection and implementation of gear design, materials, heat treatment and subsequent finishing operations.

Generally speaking, conventional press and sinter P/M technology by itself has not achieved the bending performance properties of wrought steel gears. With advanced processing techniques and the addition of shot peening, the performance gap has been significantly reduced or eliminated. Even with the value-added process of shot peening, P/M components can have strong cost advantages compared with wrought steel gearing.

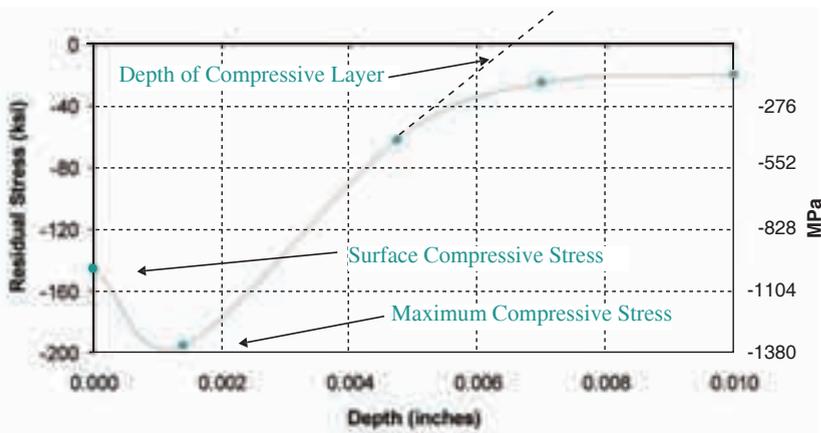


Figure 1—Typical residual stress distribution from shot-peened wrought steel.

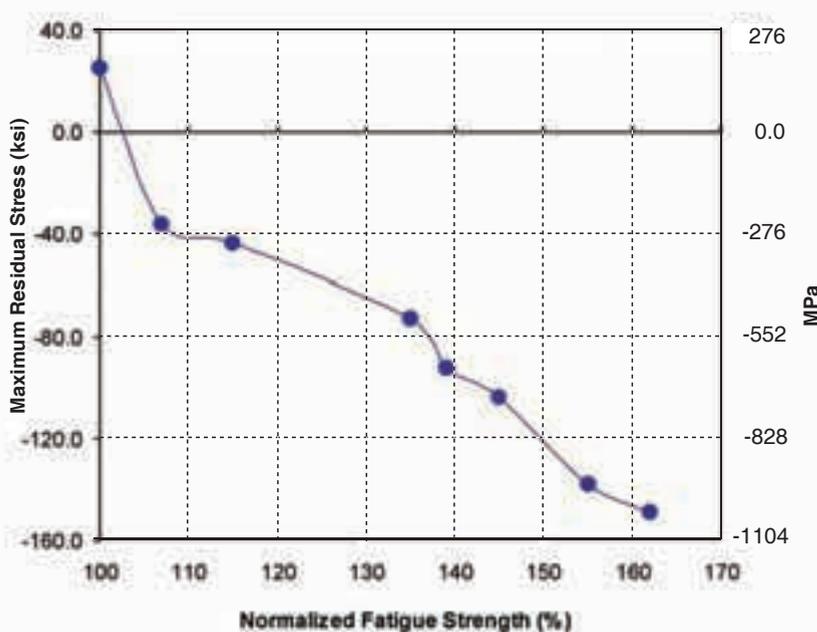


Figure 2—Increased fatigue strength correlated to increased maximum compressive stress.

P/M Technology

Usage of lower cost, higher performance manufacturing technology is heavily driven by, but not limited to, the automotive industry. Long service life, high quality requirements and continued reduction in cost are essential in manufacturing and design. The primary benefit of the P/M process is the ability to eliminate traditional manufacturing steps of machined gears to achieve near-net shapes. This net shape capability translates to significant cost savings.

Emerging P/M technology to increase load carrying capacity is primarily focused on improvements in density. Additional variations of chemistry and heat treatment are not believed to be the main limiting factors in advancing P/M gear applications.

A secondary benefit of P/M technology is very uniform and isotropic material properties. When comparing P/M steels to their wrought steel coun-

terparts, the tendency for alloy segregation may be significantly reduced. Each particle in the powder metal mix is considered a microingot. This uniformity, together with the resulting isotropy from P/M processing, results in very consistent response to heat treating and subsequent machining, grinding and finishing operations.

The P/M industry is currently advancing technologies for producing components with near-fully-dense properties. Near fully dense is defined as less than 1% residual porosity. Two of these technologies that will be discussed are powder forging and isostatic pressing.

Powder forging. This technique is used in mass produced P/M steel parts that essentially have wrought steel properties. High volume components, such as automotive transmission and engine parts, are currently being made with this process. Densities of 7.82–7.84 g/cm³ are currently possible with powder forging.

Powder forging manufacturing begins with a “green compact.” This consists of a preform that has been pressed into shape at room temperature. The preform is then heated to forging temperature and restruct in a forging tool until final density is reached.

Isostatic pressing. Isostatic pressing differs from other methods of compaction in that it is accomplished in a pressurized fluid, such as oil, water or gas. The powder is encapsulated in a flexible, sealed container (or can) that allows forming into near-net shapes of varying size and part complexity.

Hot isostatic pressing is performed in an inert gas atmosphere, most commonly nitrogen, argon or helium, contained in a pressure vessel. Both the powder (to be pressed into a formed part) and atmosphere are heated to temperatures as high as 2,300°F (1,260°C). Common pressure levels are 15,000 psi (104 MPa). Typical densities for hot isostatic pressing are 7.2–7.4 g/cm³. Some companies reportedly are able to achieve 7.5–7.8 g/cm³ with their proprietary processes.

The powder to be processed is vacuum sealed within a container that will deform plastically at elevated temperature and pressure. The pressing and sintering operations occur simultaneously in the heated pressure vessel. The container is then separated from the near-net component through methods such as leaching or machining.

Alternatively, a previously formed component of greater than 92% of theoretical density may be pressed to full density without the use of the costly encapsulation stage.

Emerging P/M technologies. Because of some economic and dimensional precision issues with powder forging and hot isostatic pressing, the focus on recent P/M technological advances is on enhance-

ments to conventional compacting, surface densification, and activated sintering technologies as means for achieving gearing-related mechanical properties that are equivalent to wrought steels (Refs. 1–2).

Enhancement through Shot Peening

Shot peening is an established technology used for inducing residual compressive stress at the surface of metal components. The residual stress is a function of the hardness/strength properties of the gear surface such that heat treatment plays a significant role in the resultant residual compressive stress profile.

For applications subject to mechanical fatigue loading, residual stresses are additive with applied stresses. The applied bending stress of a gear root is reduced by the amount of residual compression induced from the shot peen process. Shot peening modifies the residual stress distribution at the outer surface, which is usually the initiation site of a typical fatigue failure. The typical location of a bending failure for a gear is the transition/tangent point between the gear root and gear flank.

For gearing applications, the process is most commonly accomplished with steel shot media in the size range of 0.007–0.046" ϕ (0.18–1.17 mm ϕ). The shot media should be fully hardened to 55–62 HRC for maximum compressive magnitude and depth properties. Reduced hardness media (45–52 HRC) is available when tooth flank surface finish is a concern. Figure 1 shows a typical residual stress distribution of a carburized, fully hardened wrought gear steel that has been shot peened with fully hardened shot.

It should be noted that the curve does not cross the neutral axis such that the surface residual compressive stresses are balanced with sub-surface residual tensile stresses for static rebalancing. The reason is that the carburized layer (which is significantly deeper than the shot peen layer) has residual compression prior to the shot peening. The static rebalancing would occur much deeper than what is shown in Figure 1. A dashed line is shown to represent the depth of the shot peen layer by extending the curve from its positively sloping portion.

It is generally accepted for most gearing applications that bending fatigue strength improves with increasing residual compression. Figure 2 demonstrates the trend of improved bending fatigue with increasing residual compressive stress. Figure 2 is a compilation of results using carburized 20MnCr5 in single tooth pulsator tests. The gears were 8.0 mm module with 20 teeth (Ref. 3).

All results in Figure 2 are normalized to 100% on the horizontal axis. The graph shows that the highest magnitude of residual compression (~ 150 ksi or 1,035 MPa) showed an increase in fatigue strength of approximately 60% when com-

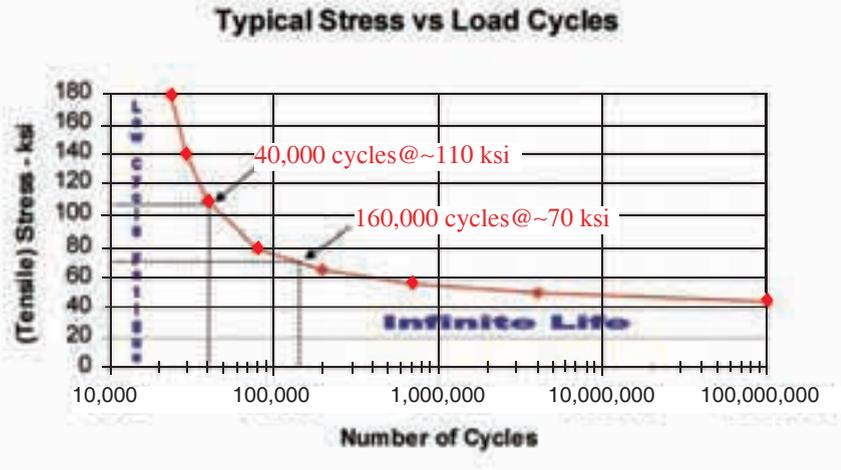


Figure 3—Typical stress vs. load cycles (S-N) curve.

pared to the baseline (100%), which had approximately 25 ksi (173 MPa) of residual tension.

As stated previously, the residual stress induced from the shot peening is heavily influenced by the properties of the material being peened. This applies to both wrought steel and P/M gears. Applying this principle to P/M components, this means that shot peening will produce more compressive stress on higher hardness and higher density P/M components, resulting in optimum load carrying properties. Lower density P/M components will still respond with compressive stress. However, they will lack the strength to retain higher magnitudes of compressive stress.

Discussion of Test Data

This section will present available test data on shot peening of powder metal gears. Several different types of data are presented, including tooth root bending fatigue data. This fatigue data was acquired using single tooth pulsator (STP) tests.

It ought to be noted that fatigue strength data from STP tests should be adjusted downward as the specific tooth (teeth) that were tested are not necessarily the weakest on the gear. In power recirculation tests, which are more applicable to real life conditions (and more expensive to perform), the weakest tooth always fails. Test results indicate a downward adjustment to ~ 80% of the STP testing improvement would be realistic (Ref. 4).

The principles of fatigue failure should also be mentioned. A typical S-N curve plots the applied stress versus number of cycles. The higher the number of cycles prior to shot peening, the greater the enhancement with the addition of shot peening. This is because shot peening lowers the net stress experienced at the surface of the gear. A reduced net stress theoretically brings the resultant stress closer to the endurance limit. Figure 3 (which does not apply to any specific material or application) demonstrates this concept.

Table 1—Tooth Root Load Carrying Capacity Via Single Tooth Pulsator Tests.

| | Wrought Reference Gear | Variation 1 (7.72 g/cm ³) MSP4.0Mo | Variation 2 (7.76 g/cm ³) MSP4.0Mo-0.1Nb |
|------------------------------|------------------------|--|--|
| Non-Peened Fatigue Strength | 131 ksi (900 MPa) | 100 ksi (685 MPa) | 108 ksi (745 MPa) |
| Shot Peened Fatigue Strength | N/A | 145 ksi (1,000 MPa) | 145 ksi (1,000 MPa) |

Note: All results are shown at 50% failure probability.

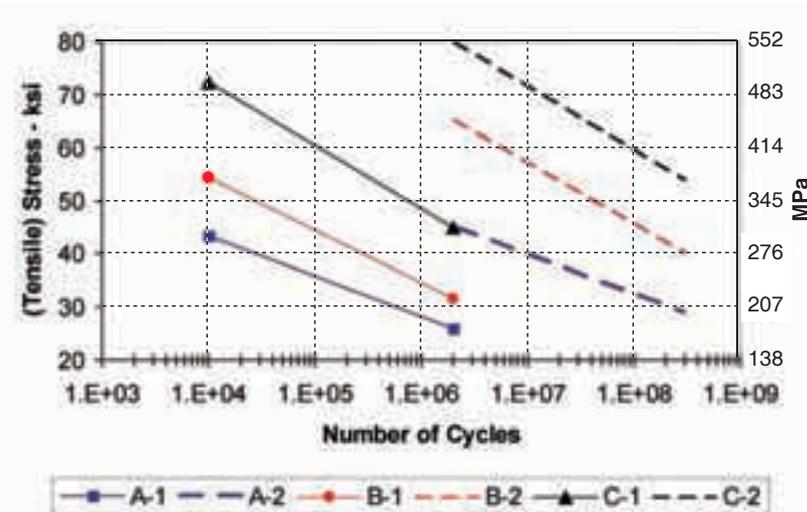


Figure 4—Fatigue life curves of Fe-2%, Cu-2.5%, Ni- with a density of 7.6 g/cm³.

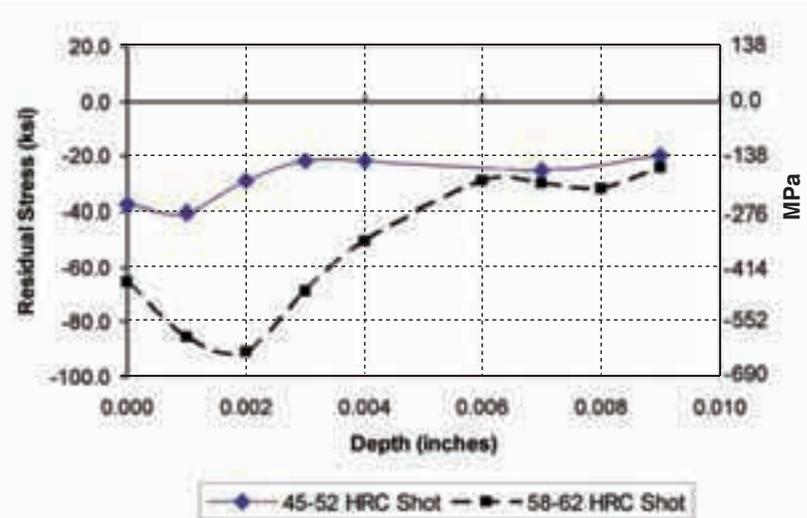


Figure 5—Residual stress plot of shot peened, carbonitrided P/M with different grades of shot hardness.

Using this principle, one could assume that if fatigue strength results were analyzed (before and after shot peening) at 500,000 cycles compared with 2 million cycles, the percent improvement would not be as significant. If fatigue strength were analyzed at 4 million cycles (compared with 2 million cycles), theoretically the fatigue strength improvement should be greater.

Case study #1. The German Federal Ministry of Education and Research tested powder metal alloys and their suitability for gearing applications. MSP4.0Mo-based powder metal gear alloys were tested against 20MnCr5 case hardened, wrought steel gears. Tooth root load carrying capacity tests were performed using single tooth pulsator tests at 2 million cycles. All gears had modules of 3.5 mm. The results are shown in Table 1 (Ref. 5).

The above test study shows a decrease of ~20% (25% & 18% respectively) when comparing the non-peened powder metal variations to the non-peened wrought reference gear. With the addition of shot peening, both examples exceeded the wrought reference gear by ~9%.

Assuming an average 104 ksi (100 & 108 ksi) fatigue strength prior to shot peening of the non-peened powder metal variations and 145 ksi with the addition of shot peening, the fatigue strength improvement is ~39%. With the previously mentioned STP-to-power recirculation adjustment, the shot peened value can be changed to ~136 ksi (a 31% improvement) for calculated service requirements.

Case study #2. Another test study was performed on P/M test bars with a machined radius acting as a stress concentration (stress factor, $K = 1.49$). The material was Fe-2%, Cu-2.5%, Ni- with a density of 7.6 g/cm³. Tests were performed under both uniform and variable loading. In addition to varying test loads, the test matrix examined various sintering treatments. Figure 4 depicts the results. Below the figure is an explanation of the curves labeled in the legend of the graph (Ref. 6).

- A-1: As Sintered, Constant Loading
- A-2: As Sintered, Variable Loading
- B-1: Sintered & Shot Peened, Constant Loading
- B-2: Sintered & Shot Peened, Variable Loading
- C-1: Sintered, Carbonitrided & Shot Peened, Constant Loading
- C-2: Sintered & Shot Peened, Variable Loading

Note: All shot peening performed to a 0.016" A intensity.

These data support the theory that harder, higher strength materials achieve better fatigue performance as they are able to retain higher magnitudes of residual compressive stress. When comparing the

constant loading conditions at 1 million cycles, the as-sintered (A-1) condition has a fatigue strength of ~ 28 ksi (193 MPa). When carbonitrided and shot peened (C-1) the fatigue strength increases to ~ 48 ksi (331 MPa), a 71% improvement.

Case study #3. Fe-Mo P/M alloy gears at a density of 7.5 g/cm³ were tested at an endurance limit of 3 million cycles. The gears were hardened to 60 HRC and shot peened (0.016" A intensity) with fully hardened 62 HRC shot. The following results are from single tooth loading (Ref. 7).

- Sintered & Case Hardened (baseline): 130 ksi (900 MPa)
- Sintered, Case Hardened & Ground: 112 ksi (770 MPa)
- Sintered, Case Hardened & Shot Peened: 149 ksi (1,030 MPa)

Shot peening improved the baseline condition by 19 ksi (~ 15%). It is worth noting that the endurance limit of the gear tooth roots that were ground decreased 18 ksi (~ 14%) from the baseline condition. It is generally believed that a smoother surface will respond better under fatigue conditions because potential crack initiation sites are assumed to be eliminated. What is sometimes neglected is the fact that grinding under certain circumstances can introduce residual tensile stresses if not properly controlled. Residual tensile stresses will act to accelerate a fatigue failure as they are additive with applied tensile stresses. It is believed this is what contributed to the decrease in fatigue life, however this was not the focus of this study.

Case study #4. Residual stress comparisons were made between shot peening a hardened surface with different grades of shot hardness. A carbonitrided, powder metal Fe-1.5% alloy (density = 7.4 g/cm³) surface was shot peened with fully hardened and regular hardness shot media. Figure 5 shows the residual stress comparison (Ref. 6).

It should be noted that both curves do not cross the neutral axis due to residual compression created from the carbonitriding process prior to shot peening. Figure 5 shows significantly more residual compression when using a harder shot media. Although bending fatigue tests for comparing the two were not available, the residual stress from the shot of higher hardness would most likely produce better results in bending fatigue.

Case study #5. Tooth root bending fatigue studies were performed using pulsator tests to compare a reference wrought gear to a powder metal gear that has a density of 7.5 g/cm³. Both gears had 3.5 mm modules, 25 teeth and were case hardened to 60 HRC. The wrought gear was a 16MnCr5 steel and the powder metal gear was Fe-3.5Mo alloy content.

In Figure 6, the powder metal gear results are depicted with the blue curves. The endurance limit

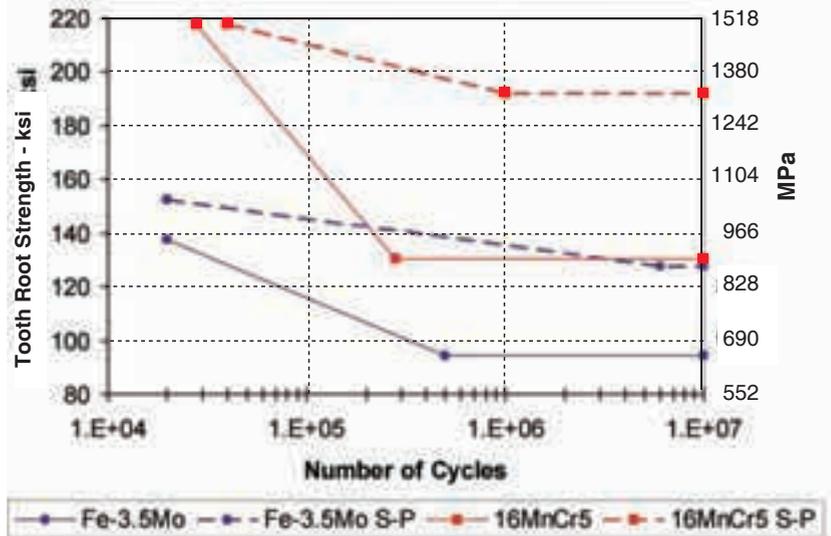


Figure 6—Fatigue life curves comparing a reference wrought steel (16MnCr5) to a powder metal (Fe-3.5Mo) of 7.5 g/cm³ density with and without shot peening.

improved ~ 35% with the addition of shot peening. The endurance limit improved from ~ 95 ksi (650 MPa) to ~ 128 ksi (880 MPa). The endurance limit of the shot peened powder metal compares very closely with the non-peened 16MnCr5 material. Shot peening was performed at 0.013" A intensity (0.32 mm A intensity) for all samples.

In Figure 6, the reference 16MnCr5 steel gear results are depicted with the red curves. The endurance limit improved ~ 45% with the addition of shot peening. The tooth root strength was ~ 131 ksi before shot peening and ~ 190 ksi after shot peening. Since the non-peened steel gear has a higher strength than the non-peened powder metal gear, the 16MnCr5 is able to retain more residual compression from shot peening, resulting in more overall improvement in fatigue strength from shot peening.

Using the previously mentioned STP-to-power recirculation adjustment, the endurance strength improvements can be changed to ~ 29% (from ~ 35%) for the powder metal gears and to ~ 36% (from ~ 45%) for the wrought steel gear (Ref. 8).

Case study #6. Via bending fatigue tests, powder forged C-0.5%, Cu-2% components with a density of 7.82 g/cm³ had a fatigue strength 27% greater than the non-peened counterparts (at 90% reliability levels) (Ref. 9). It should be noted that the components in this test were automotive connecting rods, not gears. They were included as part of this paper because they were made from high density P/M and the failure mode was bending fatigue, which is the subject of all other case studies presented in this paper.

Conclusion

The growth of the powder metal market is directly a function of the available applications

it can serve with its current cost advantages. The current market for gearing covers a broad spectrum from non-critical, low load applications to critical, high load applications. The P/M gear market is naturally limited by its mechanical performance capability.

Recent improvements in the manufacturing technologies of powder metal have allowed it to be considered for higher strength gear applications. Higher bending strength P/M applications are highly predicated on higher densities and subsequent heat treating and metal finishing operations.

Typical high load gear applications are susceptible to bending fatigue failure in the gear tooth root. Shot peening is a recognized process to induce residual compressive stress. It is also highly dependent on P/M density and heat treatment. Test data were presented to support the consideration of P/M bending fatigue strength improvements through the use of shot peening. P/M gears (even with shot peening) offer significant cost advantages over their wrought steel counterparts. 

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